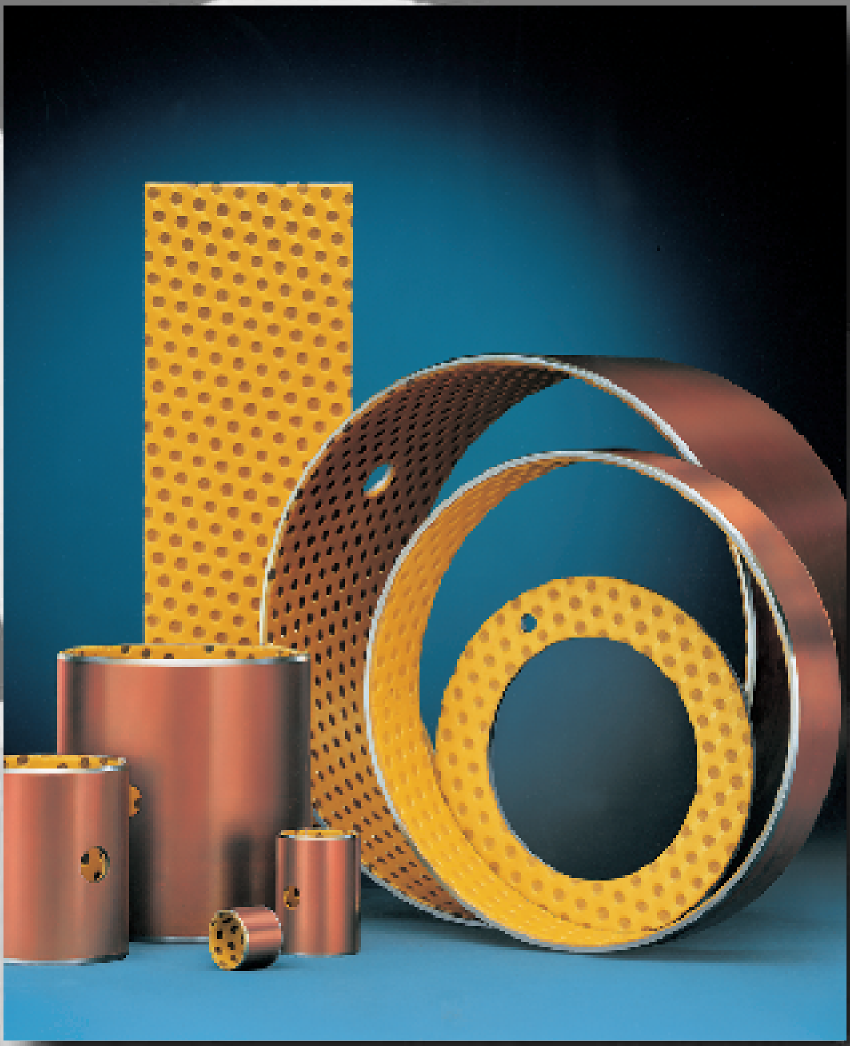
GLACIER GARLOCK BEARINGS

DX

Marginally lubricated



Designer´s Handbook

Quality

All the products described in this handbook are manufactured under DIN ISO 9001/2 or QS 9000 approved quality management systems.

Formula Symbols and Designations

Formula Symbol	Unit	Designation
a_B	-	Bearing size factor
a_E	-	High load factor
a_Q	-	Speed/Load factor
a_S	-	Surface finish factor
a_T	-	Temperature application factor
B	mm	Nominal bush width
C	1/min	Dynamic load frequency
C_D	mm	Installed diametral clearance
C_{Dm}	mm	Diametral clearance machined
C_T	-	Total number of dynamic load cycles
C_i	mm	ID chamfer length
C_o	mm	OD chamfer length
D_H	mm	Housing Diameter
D_i	mm	Nominal bush/thrust washer ID
$D_{i,a}$	mm	Bush ID when assembled in housing
$D_{i,a,m}$	mm	Bush ID assembled and machined
D_J	mm	Shaft diameter
D_o	mm	Nominal bush/thrust washer OD
d_D	mm	Dowel hole diameter
d_L	mm	Oil hole diameter
d_P	mm	Pitch circle diameter for dowel hole
F	N	Bearing load
F_i	N	Insertion force
f	-	friction
L	mm	Strip length
L_H	h	Bearing service life
L_{RG}	h	Relubrication interval
N	1/min	Rotational speed
N_{osz}	1/min	Oscillating movement frequency
$\bar{p}$	N/mm^2	Specific load
$\bar{p}_{lim}$	N/mm^2	Specific load limit
$\bar{p}_{sta,max}$	N/mm^2	Maximum static load
$\bar{p}_{dyn,max}$	N/mm^2	Maximum dynamic load
Q	-	Total number of cycles
R	-	Number of lubrication intervals
R_a	µm	Surface roughness (DIN 4768, ISO/DIN 4287/1)
S_3	mm	Bush wall thickness
S_S	mm	Strip thickness
S_T	mm	Thrust washer thickness
T	°C	Temperature
T_a	mm	Depth of Housing Recess
T_{amb}	°C	Ambient temperature
T_{max}	°C	Maximum temperature
T_{min}	°C	Minimum temperature
U	m/s	Sliding speed
u	-	speed factor
W	mm	Strip width
W_u	mm	Maximum usable strip width
α_1	$1/10^6K$	Coefficient of linear thermal expansion parallel to surface
α_2	$1/10^6K$	Coefficient of linear thermal expansion normal to surface
σ_c	N/mm^2	Compressive Yield strength
λ	W/mK	Thermal conductivity
φ	°	Angular displacement
η	Ns/mm^2	Dynamic Viscosity
Z_T	-	Total number of osscillating movements

Content

1 Introduction

The purpose of this handbook is to provide comprehensive technical information on the characteristics of DX bearings. The information given, permits designers to establish the correct size of bearing required and the expected life and performance. Glacier Garlock Bearings Research and Development services are available to assist with unusual design problems.

Complete information on the range of DX standard stock products is given together with details of other DX products.

Glacier Garlock Bearings is continually refining and extending its experimental and theoretical knowledge and, therefore, when using this brochure it is always worthwhile to contact the Company should additional information be required.

Customers are advised to carry out prototype testing wherever possible.

1.1 Characteristics and Advantages

- **DX provides maintenance free operation**
- **DX has a high $\bar{p}U$ capability**
- **DX exhibits low wear rate**
- **Seizure resistant**
- **Suitable for temperatures from -40 to +120 °C**
- **High static and dynamic load capacity**
- **Good frictional properties**
- **No water absorption and therefore dimensionally stable**
- **Compact and light**
- **Suitable for rotating, oscillating, reciprocating and sliding movements**
- **DX bearings are prefinished and require no machining after assembly**

2 Structure

DX is a composite bearing material developed specifically to operate with marginal lubrication and consists of three bonded layers: a steel backing strip and a sintered porous bronze matrix, impregnated and overlaid with a pigmented acetal copolymer bearing material.

The steel backing provides mechanical strength and the bronze interlayer provides a strong mechanical bond for the lining. This construction promotes dimensional stability and improves thermal conductivity, thus reducing the temperature at the bearing surface.

DX is designed for use with grease lubrication and the bearing surface is normally provided with a uniform pattern of indents These serve as a reservoir for the grease and are designed to provide the optimum distribution of the lubricant over the bearing surface.

Fig. 1: DX-microsection

2.1 Basic Forms

Standard Components available from stock

These products are manufactured to International, National or Glacier Garlock Bearings standard designs.

Metric and Imperial Sizes

- Cylindrical Bushes
  - PM pre finished metric range, not machinable in situ, for use with standard journals finished to h6-h8 limits.
  - MB machinable metric range, with an allowance for machining in situ.
  - Machinable inch range for use as supplied or after machining in situ.
- Thrust Washers
- Strip Material

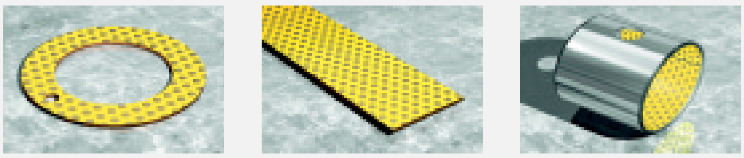


Fig. 2: Standard components

Non Standard Components not available from stock

These products are manufactured to customers' requirements with or without Glacier Garlock Bearings recommendations, and include for example

- Modified Standard Components
- Half Bearings
- Flat Components
- Pressings
- Stampings

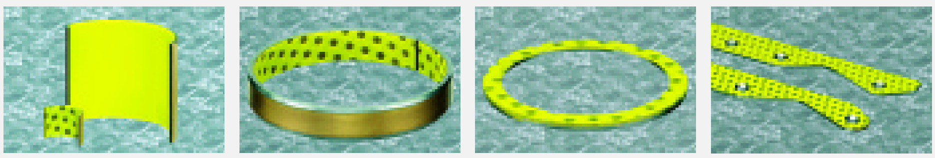


Fig. 3: Non standard components

3 Properties

3.1 Physical Properties

	Characteristic	Symbol	Value DX	Unit	Comments
Physical Properties	Thermal Conductivity	λ	52	W/mK	
	Coefficient of linear thermal expansion :				
	parallel to surface	α_1	11	$1/10^6$K	
	normal to surface	α_2	29	$1/10^6$K	
	Maximum Operating Temperature	T_{max}	120	°C	
	Minimum Operating Temperature	T_{min}	–40	°C	
Mechanical Properties	Compressive Yield Strength	σ_c	380	N/mm²	measured on disc 5 mm diameter x 2.45 mm thick.
	Maximum Load				
	Static	$\bar{p}_{sta,max}$	140	N/mm²	
	Dynamic	$\bar{p}_{dyn,max}$	140	N/mm²	
Electrical Properties	Volume resistivity of acetal lining		10^{15}	Ωcm	

Table 1: Properties of DX

3.2 Chemical Properties

The following table provides an indication of the resistance of DX to various chemical media. It is recommended that the chemical resistance is confirmed by testing if possible.

+	**Satisfactory:** Corrosion damage is unlikely to occur.
o	**Acceptable:** Some corrosion damage may occur but this will not be sufficient to impair either the structural integrity or the tribological performance of the material.
-	**Unsatisfactory:** Corrosion damage will occur and is likely to affect either the structural integrity and/or the tribological performance of the material.

	Chemical	%	°C	Rating
Strong Acids	**Hydrochloric Acid**	5	20	-
	Nitric Acid	5	20	-
	Sulphuric Acid	5	20	-
Weak Acids	**Acetic Acid**	5	20	-
	Formic Acid	5	20	-
Bases	**Ammonia**	10	20	o
	Sodium Hydroxide	5	20	o
Solvents	**Acetone**		20	+
	Carbon Tetrachloride		20	+
Lubricants and fuels	**Paraffin**		20	+
	Gasolene		20	+
	Kerosene		20	+
	Diesel fuel		20	+
	Mineral Oil		70	o
	HFA-ISO46 High Water fluid		70	o
	HFC-Water-Glycol		70	o
	HFD-Phosphate Ester		70	+
	Water		20	o
	Sea Water		20	-

Table 2: Chemical resistance of DX

4 Lubrication

4.1 Choice of Lubricant

DX must be lubricated, the choice of lubricant depending upon pU and sliding speed and the stability of the lubricant under the operating conditions.

+	**Recommended**
o	**Satisfactory**
-	**Not recommended**
NA	**Data not available**

Manufacturer	Grade	Type		Rating
BP	Energrease LS2	Mineral	Lithium Soap	+
	Energrease LT2	Mineral	Lithium Soap	+
	Energrease FGL	Mineral	Non Soap	o
	Energrease GSF	Synthetic	NA	o
Century	Lacerta ASD	Mineral	Lithium/Polymer	o
	Lacerta CL2X	Mineral	Calcium	-
Dow Corning	Molykote 55M	Silicone	Lithium Soap	o
	Molykote PG65	PAO	Lithium Soap	+
	Molykote PG75	Synthetic/Mineral	Lithium Soap	+
	Molykote PG602	Mineral	Lithium Soap	o
Elf	Rolexa.1	Mineral	Lithium Soap	+
	Rolexa.2	Mineral	Lithium Soap	o
	Epexelf.2	Mineral	Lithium/Calcium Soap	o
Esso	Andok C	Mineral	Sodium Soap	o
	Andok 260	Mineral	Sodium Soap	o
	Cazar K	Mineral	Calcium Soap	-
Mobil	Mobilplex 47	Mineral	Calcium Soap	o
	Mobiltemp 1	Mineral	Non Soap	+
Rocol	BG622	White Mineral	Calcium Soap	o
	Sapphire	Mineral	Lithium Complex	o
	White Food Grease	White Oil	Clay	-
Shell	Albida R2	Mineral	Lithium Complex	+
	Axinus S2	Mineral	Lithium	o
	Darina R2	Mineral	Inorganic Non Soap	+
	Stamina U2	Mineral	Polyurea	o
	Tivela A	Synthetic	NA	+
Sovereign	Omega 77	Mineral	Lithium	o
	Omega 85	Mineral	Polyurea	-
Tom Pac	Tom Pac	NA	NA	o
Total	Aerogrease	Synthetic	NA	+
	Multis EP2	NA	Lithium	-

Table 3: Performance of greases

4 Lubrication

Grease

This is the recommended method of lubrication. The performance ratings of different types of grease are indicated in Table 3. For environmental temperatures above 50°C the grease should contain an anti-oxidant additive. Greases containing EP additives or significant additions of graphite or MoS_2 are not generally recommended for use with DX.

Oil

DX is not generally suitable for use with hydrocarbon oils operating above 115 °C. At these temperatures oxidation of the oil may produce a low concentration of labile residues, acid or free radical, which will cause depolymerisation of the DX acetal copolymer bearing lining. Such oxidation can also occur after prolonged periods at lower temperatures. In practice, this means that DX is not recommended for use with recirculating oil systems or bath systems where sump temperature of 70 °C or greater are possible.

Non lubricating fluids

Care must be taken when using DX with non lubricating fluids as indicated below.

Water

DX is only suitable for operation in water when the load and speed permit full hydrodynamic conditions to be established (see Fig. 7).

Water-Oil Emulsion

DX is suitable for use with 95/5 water/oil emulsions, however initial operation with pure oil or grease is recommended before transferring to emulsion.

Shock-Absorber Oils

DX is not compatible with shock-absorber oils at operating temperature.

Petrol

With petrol as a lubricant at a $\bar{p}U$ factor of 0.21 N/mm^2 x m/s the wear rate of DX has been found to be about 4-5 times greater than that of an initially greased bearing under the same $\bar{p}U$ conditions.

Kerosene and Polybutene

The wear rate of DX with these fluids has been found to be equivalent to that obtained with a light hydrocarbon oil.

Other Fluids

Polyester, polyethylene glycol and polyglycol lubricants give similar wear rates with DX to light hydrocarbon oil. With the glycol fluids however the operating temperature must not exceed 80°C because of the possibility of attack of the acetal lining of DX by these fluids.

In general, the fluid will be acceptable if it does not chemically attack the acetal lining or the porous bronze interlayer. Chemical resistance data are given in Table 2.

Where there is doubt about the suitability of a fluid, a simple test is to submerge a sample of DX material in the fluid for two to three weeks at 15-20°C above the operating temperature. The following will usually indicate that the fluid is not suitable for use with DX.

- A significant change in the thickness of the DX material,
- A visible change in the bearing surface from polished to matt.
- A visible change in the microstructure of the bronze interlayer

4.2 Friction

Lubricated DX bearings show negligible 'stick-slip' and provide smooth sliding between adjacent surfaces. The coefficient of friction of lubricated DX depends upon the actual operating condi-tions as indicated in section 4.3. Where frictional characteristics are critical to a design they should be established by prototype testing.

4.3 Lubricated Environments

The following sections describe the basics of lubrication and provide guidance on the application of DX in such environments.

Lubrication

There are three modes of lubricated bearing operation which relate to the thickness of the developed lubricant film between the bearing and the mating surface.

These three modes of operation depend upon:

- Bearing dimensions
- Clearance
- Load and Speed
- Lubricant Viscosity and Flow

Hydrodynamic lubrication

Characterised by:

- Complete separation of the shaft from the bearing by the lubricant film
- Very low friction and no wear of the bearing or shaft since there is no contact
- Coefficients of friction of 0.001 to 0.01

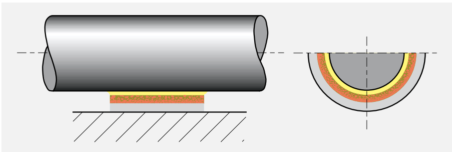


Fig. 4: Hydrodynamic lubrication

Hydrodynamic conditions occur when

(4.3.1) [N/mm²]

$$\bar{p} \leq \frac{U \cdot \eta}{7.5} \cdot \frac{B}{D_i}$$

Mixed film lubrication

Characterised by:

- Combination of hydrodynamic and boundary lubrication.
- Part of the load is carried by localised areas of self pressurised lubricant and the remainder supported by boundary lubrication.
- Friction and wear depend upon the degree of hydrodynamic support developed.
- DX provides low friction and high wear resistance to support the boundary lubricated element of the load.

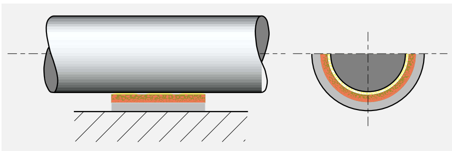


Fig. 5: Mixed film lubrication

Boundary lubrication

Characterised by:

- Rubbing of the shaft against the bearing with virtually no lubricant separating the two surfaces.
- Bearing material selection is critical to performance.
- Shaft wear is likely due to contact between bearing and shaft.
- The excellent properties of DX material minimises wear under these conditions.
- The dynamic coefficient of friction with DX is typically 0.02 to 0.1 under boundary lubrication conditions.
- The static coefficient of friction with DX is typically 0.03 to 0.15 under boundary lubrication conditions.

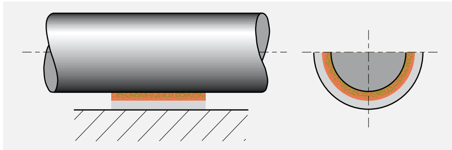


Fig. 6: Boundary lubrication

4.4 Characteristics of Fluid Lubricated DX Bearings

DX is particularly effective in the most demanding of lubricated applications where full hydrodynamic operation cannot be maintained, for example:

- **High load conditions**
  In highly loaded applications operating under boundary or mixed film conditions DX shows excellent wear resistance and low friction.
- **Start up and shut down under load**
  With insufficient speed to generate a hydrodynamic film the bearing will operate under boundary or mixed film conditions.
  - DX minimises wear
  - DX requires less start up torque than conventional metallic bearings.
- **Sparse lubrication**
  Many applications require the bearing to operate with less than the ideal lubricant supply, typically with splash or mist lubrication only.
  DX requires significantly less lubricant than conventional metallic bearings.

4.5 Design Guidance for Fluid Lubricated Applications

Fig. 7, Page 11 shows the three lubrication regimes discussed above plotted on a graph of sliding speed vs the ratio of specific load to lubricant viscosity.

In order to use Fig. 7

- Using the formulae in Section 5
  - Calculate the specific load $\bar{p}$
  - Calculate the shaft surface speed(U)
- Using the viscosity temperature relationships presented in Table 4.
  - Determine the viscosity in centipoise of the lubricant.

Note:

Viscosity is a function of operating temperature. If the operating temperature of the fluid is unknown, a provisional temperature of 25 °C above ambient can be used.

Area 1 of Fig. 7

The bearing will operate with boundary lubrication.

The $\bar{p}U$ factor will be the major determinant of bearing life.

DX bearing performance can be estimated from the following:

Calculate Effective $\bar{p}U$ Factor from Section 5.8

If $e\bar{p}U/\eta \leq 0.2$ then

(4.5.1)
$$L_H = \frac{2000}{\left(\frac{e\bar{p}U}{\eta}\right)^{0.5}} \cdot a_Q \cdot a_T \cdot a_S \quad [h]$$

If $0.2 < e\bar{p}U/\eta \leq 1.0$ then

(4.5.1)
$$L_H = \frac{1000}{\left(\frac{e\bar{p}U}{\eta}\right)} \cdot a_Q \cdot a_T \cdot a_S \quad [h]$$

If $e\bar{p}U/\eta > 1.0$ then

(4.5.1)
$$L_H = \frac{1000}{\left(\frac{e\bar{p}U}{\eta}\right)^{2}} \cdot a_Q \cdot a_T \cdot a_S \quad [h]$$

$e\bar{p}U$ see (5.8.2), Page 19

Area 2 of Fig. 7

The bearing will operate with mixed film lubrication.

$\bar{p}U$ factor is no longer a significant parameter in determining the bearing life.

DX bearing performance will depend upon the nature of the fluid and the actual service conditions.

Area 3 of Fig. 7

The bearing will operate with hydrodynamic lubrication. Bearing wear will be determined only by the cleanliness of the lubricant and the frequency of start up and shut down.

Area 4 of Fig. 7

- These are the most demanding operating conditions.
- The bearing is operated under either high speed or high bearing load to viscosity ratio, or a combination of both.
- These conditions may cause
  - excessive operating temperature
  - and/or high wear rate.
- Bearing performance may be improved:
  - by use of unindented DX lining
  - by the addition of one or more grooves to the bearing
  - by shaft surface finish < 0.05 [µm Ra].

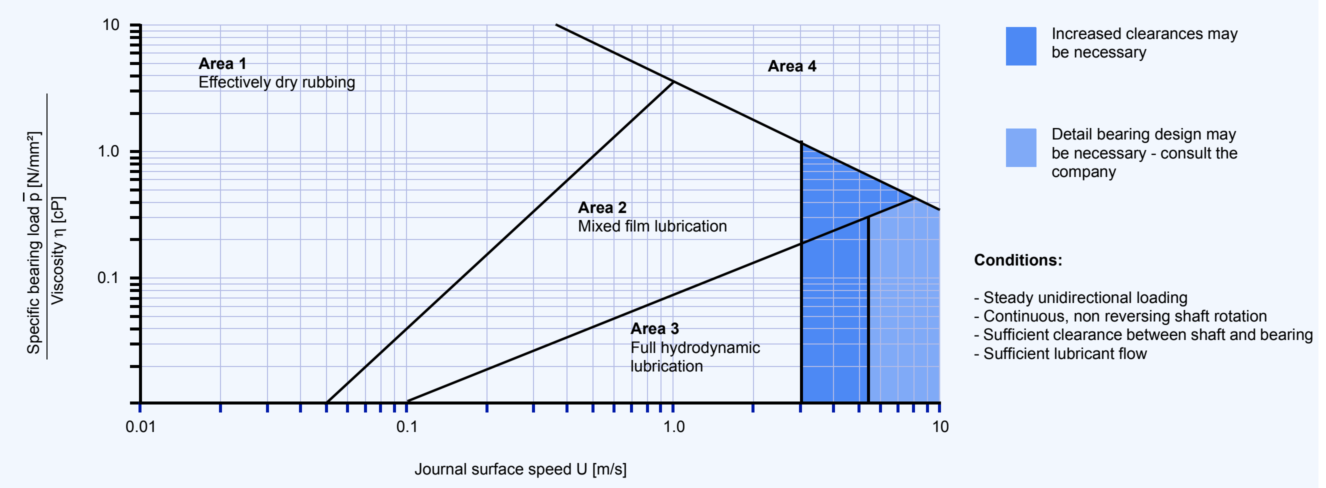


Fig. 7: Design guide for lubricated application

	cP														
Temperature [°C]	**0**	**10**	**20**	**30**	**40**	**50**	**60**	**70**	**80**	**90**	**100**	**110**	**120**	**130**	**140**
Lubricant															
ISO VG 32	310	146	77	44	27	18	13	9.3	7.0	5.5	4.4	3.6	3.0	2.5	2.2
ISO VG 46	570	247	121	67	40	25	17	12	9.0	6.9	5.4	4.4	3.6	3.0	2.6
ISO VG 68	940	395	190	102	59	37	24	17	12	9.3	7.2	5.8	4.7	3.9	3.3
ISO VG 100	2110	780	335	164	89	52	33	22	15	11.3	8.6	6.7	5.3	4.3	3.6
ISO VG 150	3600	1290	540	255	134	77	48	31	21	15	11	8.8	7.0	5.6	4.6
Diesel oil	4.6	4.0	3.4	3.0	2.6	2.3	2.0	1.7	1.4	1.1	0.95				
Petrol	0.6	0.56	0.52	0.48	0.44	0.40	0.36	0.33	0.31						
Kerosene	2.0	1.7	1.5	1.3	1.1	0.95	0.85	0.75	0.65	0.60	0.55				
Water	1.79	1.30	1.0	0.84	0.69	0.55	0.48	0.41	0.34	0.32	0.28				

Table 4: Viscosity data

4.6 Wear Rate and Relubrication Intervals with Grease lubrication

At specific bearing loads below 100 N/mm^2 a grease lubricated DX bearing shows only small bedding-in wear of about 0.0025 mm. This is followed by little wear during the early part of the bearing life until the lubricant becomes exhausted and the wear rate increases. If the bearing is regreased before the rate of wear starts to increase rapidly the material will continue to function satisfactorily with little wear. Fig. 8 shows the typical wear pattern.

Under specific loads above 100 N/mm^2 the initial bedding-in wear is greater, typically about 0.025 mm, followed by a decreasing wear rate until the bearing exhibits a similar wear/life relationship to that shown in Fig. 8.

The useful life of the bearing is limited by wear in the loaded area. If this wear exceeds 0.15mm the grease capacity of the indents is reduced and more frequent regreasing of the bearing will be required.

Fretting Wear

Oscillating movements of less than the dimensions of the indent pattern may cause localised wear of the mating surface after prolonged usage. This will result in the indent pattern becoming transferred onto the mating surface in contact with the DX bearing and may also give rise to fretting corrosion damage. In this situation DS™ material should be considered as an alternative to DX.

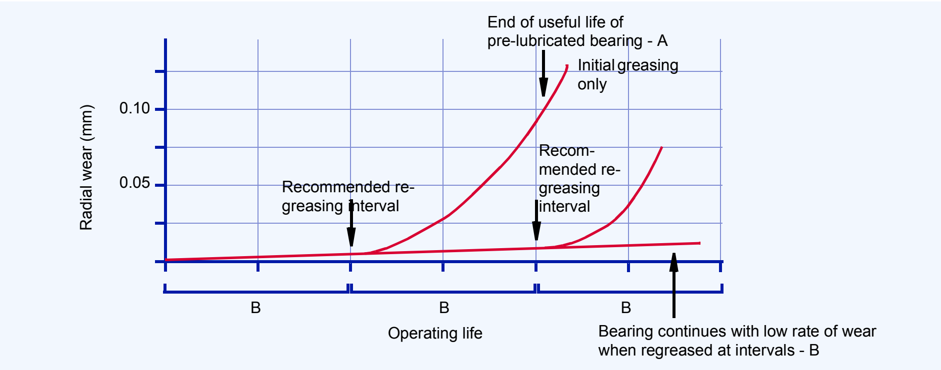


Fig. 8: Typical wear of DX

5 Design Factors

The main parameters when determining the size or calculating the service life for a DX bearing are:

- Specific Load Limit $\bar{p}_{lim}$ [N/mm^2]
- $\bar{p}$U Factor [N/mm^2 x m/s]
- Mating surface roughness R_a [µm]
- Mating surface material
- Temperature T [°C]
- Other environmental factors eg. housing design, dirt, lubrication.

5.1 Specific Load

The specific load $\bar{p}$ is defined as the working load divided by the projected area of the bearing and is expressed in N/mm^2.

Bushes

(5.1.1) [N/mm²]

$$\bar{p} = \frac{F}{D_i \cdot B}$$

Thrust Washers

(5.1.2) [N/mm²]

$$\bar{p} = \frac{4F}{\pi \cdot (D_o^2 - D_i^2)}$$

Slide Plates

(5.1.3) [N/mm²]

$$\bar{p} = \frac{F}{L \cdot W}$$

Specific Load Limit

The maximum load which can be applied to a DX bearing can be expressed in terms of the Specific Load Limit, which depends on the type of the loading and lubrication It is highest under steady loads. The values of Specific Load Limit specified in Table 5 assume good alignment between the bearing and mating surface.

The Specific Load Limit for DX reduces for bearing operating temperatures in excess of 40 °C, falling to about half the values given in Table 5 for temperatures above 100 °C.

Conditions of dynamic load or oscillating movement which produce fatigue stress in the bearing result in a reduction in the permissible Specific Load Limit (Fig. 9, Page 14).

Load	Operating condition	Lubrication	$\bar{p}_{lim}$
Steady	Intermittent or very slow (below 0.01 m/s) continuous rotation or oscillating motion	Grease or oil	140
Steady	Continuous rotation or oscillating motion	Grease or oil (boundary lubrication)	70
Steady or dynamic	Continuous rotation or oscillating motion	Oil (hydrodynamic lubrication)	45

Table 5: Specific load limit $\bar{p}_{lim}$ for DX

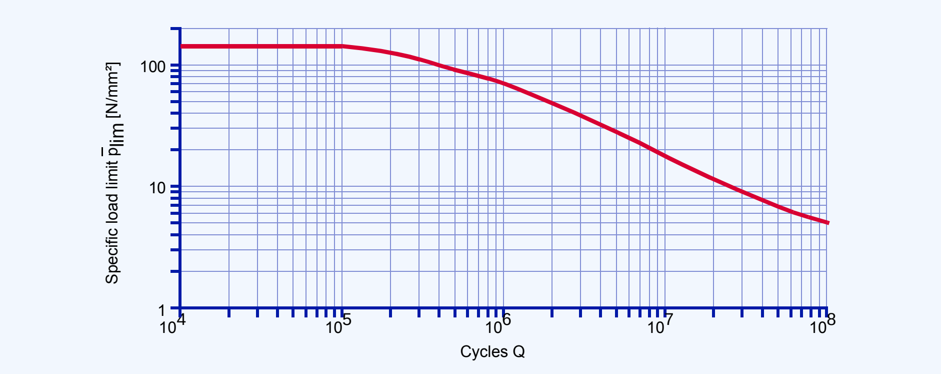


Fig. 9: DX specific load limits $\bar{p}_{lim}$ under dynamic loads or oscillating conditions

5.2 Sliding Speed

The sliding speed U [m/s] is calculated as follows:

Continuous Rotation

Bushes

(5.2.1) [N/mm²]

$$U = \frac{D_i \cdot \pi \cdot N}{60 \cdot 10^3}$$

Thrust Washers

(5.2.2) [N/mm²]

$$U = \frac{\frac{D_o + D_i}{2} \cdot \pi \cdot N}{60 \cdot 10^3}$$

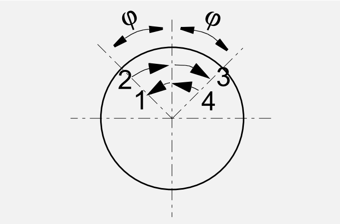


Fig. 10: Oscillating cycle φ

Oscillating Movement

Bushes

(5.2.3) [N/mm²]

$$U = \frac{D_i \cdot \pi}{60 \cdot 10^3} \cdot \frac{4\varphi \cdot N_{osz}}{360}$$

Thrust Washers

(5.2.4) [N/mm²]

$$U = \frac{\frac{D_o + D_i}{2} \cdot \pi}{60 \cdot 10^3} \cdot \frac{4\varphi \cdot N_{osz}}{360}$$

The maximum permissible effective $\bar{p}U$ factor ($e\bar{p}U$ factor) for grease lubricated DX bearings is dependent upon the sliding speed as shown in Fig. 11. For sliding speeds in excess of 2.5 m/s continuous oil lubrication is recommended.

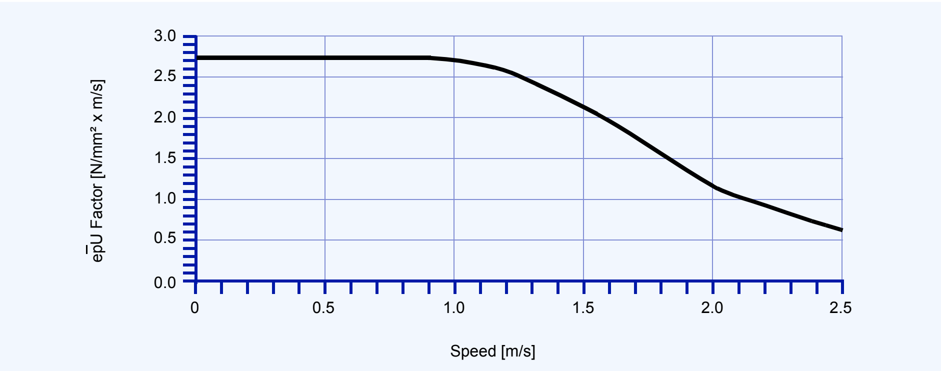


Fig. 11: Maximum $e\bar{p}U$ factor for grease lubrication

5.3 $\bar{p}U$ Factor

The useful operating life of a DX bearing is governed by the $\bar{p}U$ factor, which is calculated as follows:

(5.3.1) [N/mm² x m/s]

$$\bar{p}U = \bar{p} \cdot U$$

5.4 Load

In addition to its contribution to the $\bar{p}U$ factor the type and direction of the applied load also affects the performance of a DX bearing. This is accomodated in the calculation of the bearing service life by the speed/load application factor a_Q shown in Figs. 15-17.

Type of Load

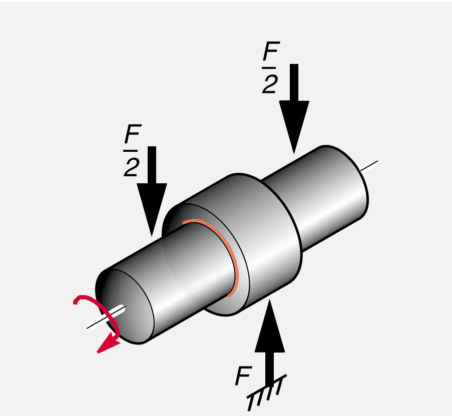


Fig. 12: Steady load, vertically downwards, bush stationary, shaft rotating. Lubricant drains to loaded area

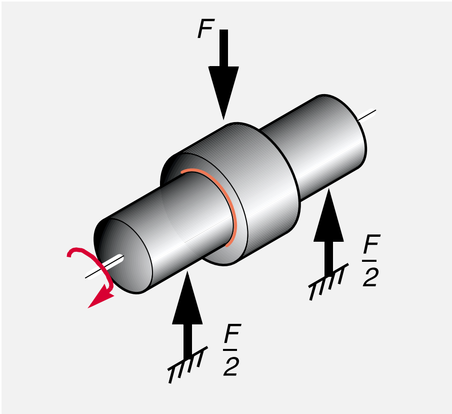


Fig. 13: Steady load, vertically upwards, bush stationary, shaft rotating. Lubricant drains away from loaded area

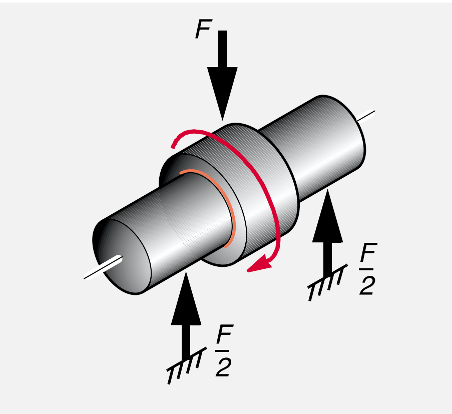


Fig. 14: Rotating load, shaft stationary, bush rotating

5 Design Factors

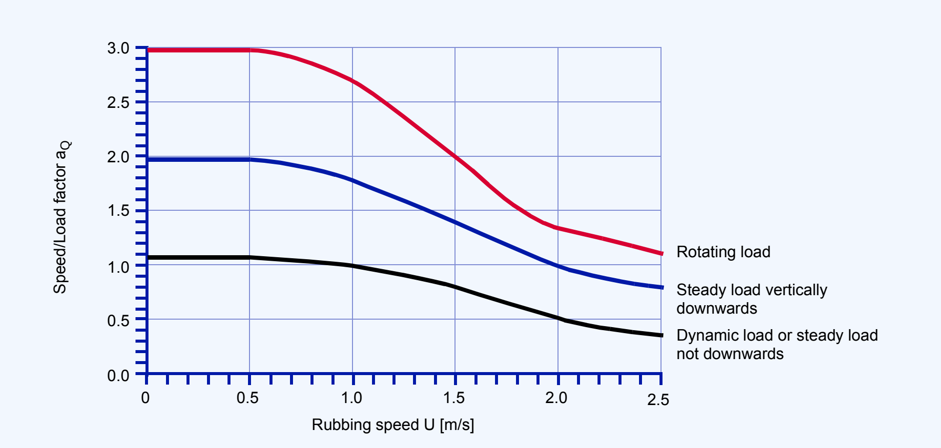


Fig. 15: Speed/Load factor a_Q for MB range bushes - unmachined

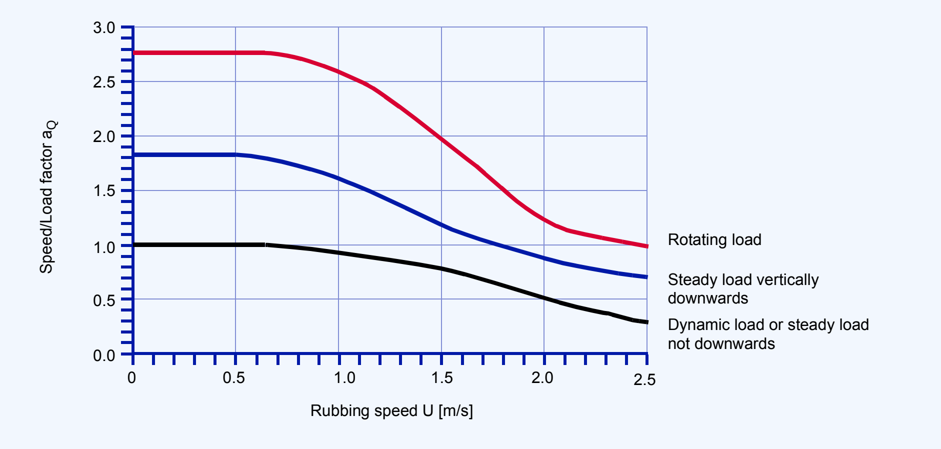


Fig. 16: Speed/Load factor a_Q for PM range and MB range bushes - machined

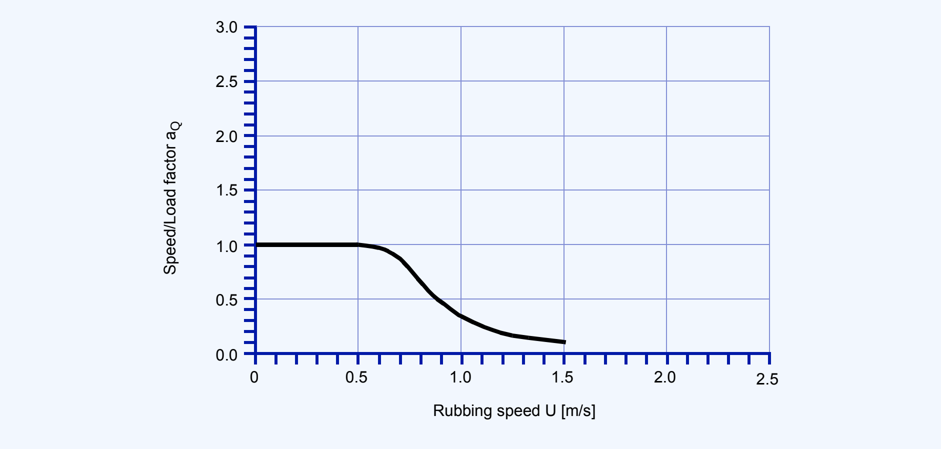


Fig. 17: Speed/Load factor a_Q for thrust washers

Note: a_Q = 1 for slideways

5.5 Temperature

The useful life of a DX bearing depends upon the operating temperature. The performance of grease lubricated DX decreases at bearing temperatures above 40°C. This loss of performance is related to both material and lubricant effects.

For a given $\bar{p}U$ Factor the operating temperature of the bearing depends upon the temperature of the surrounding environment and the heat dissipation properties of the housing.

In calculating the service life of DX these effects are accomodated by the application factor a_T shown in Fig. 18.

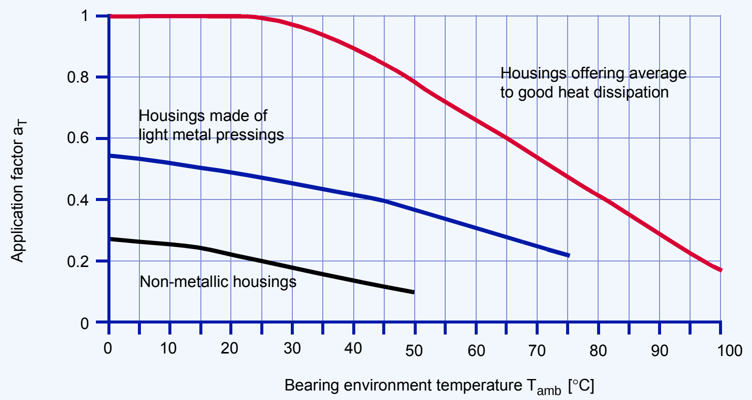


Fig. 18: DX application factor a_T

5.6 Mating Surface

The wear rate of DX is strongly dependent upon the roughness of the mating counterface. For optimum bearing performance the mating surface should be ground to better than 0.4 µm R_a. This effect is accomodated by the mating surface finish application factor a_S shown in Fig. 19.

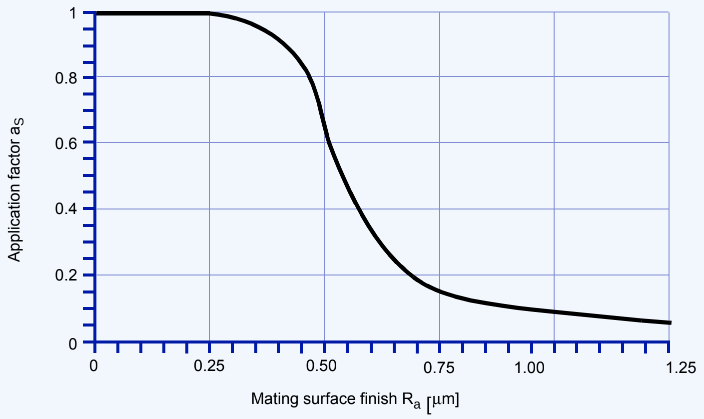


Fig. 19: DX application factor a_S

5 Design Factors

5.7 Bearing Size

Frictional heat generated at the bearing surface and dissipated through the shaft and housing depends both on the operating conditions (i.e. $\bar{p}U$ factor) and the bearing size.

For a give $\bar{p}U$ condition a large bearing will run hotter than a smaller bearing. The bearing size factor a_B shown in Fig. 20 takes account of this effect.

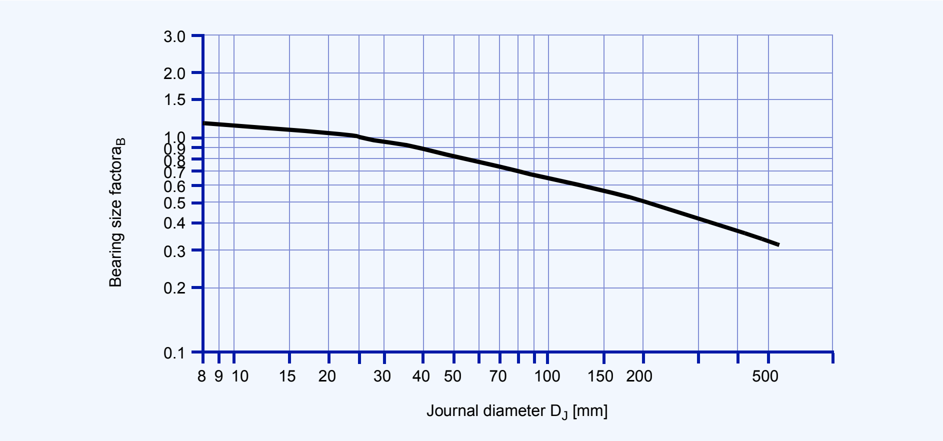


Fig. 20: Bearing size factor a_B

Note: a_B = 1 for slideways

5.8 Estimation of Bearing Service Life with Grease Lubrication

Calculation Parameters

Bushes		Thrust Washers		Slide Plates		Unit
Bearing diameter	D_i	Bearing outside diameter	D_o	Strip Length	L	[mm]
Bearing width	B	Bearing inside diameter	D_i	Strip Width	W	[mm]

Operating Conditions

Load	F	[N]
Rotational Speed (Continuous)	N	[1/min]
Oscillating Frequency	N_{osz}	[1/min]
Angular movement about mean position	φ	[°]
Specific Load Limit	see Table 5, Page 13	[N/mm²]
Application Factor a_Q	see Fig. 15-17, Page 16	[–]
Application Factor a_T	see Fig. 18, Page 17	[–]
Application Factor a_S	see Fig. 19, Page 17	[–]
Bearing Size Factor a_B	see Fig. 20, Page 18	[–]

Calculate $\bar{p}$ from the equations in 5.1 on Page 13.

Calculate U from the equations in 5.2 on Page 14.

Calculate $\bar{p}U$ from the equation in 5.3 on Page 15.

Calculate High Load Factor a_E

(5.8.1) [–]

$$a_E = \frac{\bar{p}_{lim}}{\bar{p}_{lim} - \bar{p}}$$

$\bar{p}$lim see Table 5, Page 13

Note:

If $a_E > 10000$, or $a_E < 0$, the bearing is overloaded.

Calculate Effective $\bar{p}U$ Factor $e\bar{p}U$

(5.8.2) [–]

$$e\bar{p}U = \frac{a_E \cdot \bar{p}U}{a_B}$$

Note:

Check that $e\bar{p}U$ is less than limit set in Fig. 11 for the sliding speed U. If NOT, increase the bearing length or use continuous lubrication.

Estimate Bearing Life

If $e\bar{p}U < 1.0$ then

(5.8.3) [h]

$$L_H = \frac{3000}{e\bar{p}U} \cdot a_Q \cdot a_T \cdot a_S$$

If $e\bar{p}U > 1.0$ then

(5.8.4) [h]

$$L_H = \frac{3000}{(e\bar{p}U)^{2.4}} \cdot a_Q \cdot a_T \cdot a_S$$

Estimate Re-greasing Interval

(5.8.5) [h]

$$L_{RG} = \frac{L_H}{2}$$

Oscillating Motion

Calculate number of cycles

(5.8.6) [–]

$$Z_T = L_{RG} \cdot N_{osz} \cdot 60 \cdot (R + 2)$$

Dynamic Loads

Calculate number of cycles

(5.8.7) [–]

$$C_T = L_{RG} \cdot C \cdot 60 \cdot (R + 2)$$

where R = Number of times bearing is regreased during total life required.

Check that Z_T (or C_T) is less than the total number of cycles Q given in Fig. 9 for actual bearing specific load $\bar{p}$.

If Z_T (or C_T) > Q then life will be limited by fatigue after Q cycles.

If Z_T (or C_T) < Q then life will be limited by wear after Z_T cycles.

If the estimated life or total cycles are insufficient or the regreasing intervals are too frequent, increase the bearing length or diameter, or consider drip feed or continuous oil lubrication, the quantity to be established by test.

5.9 Worked Examples

PM cylindrical Bush

Given:			
Load Details	Steady Load	Inside Diameter D_i	40 mm
	Direction: down	Length B	30 mm
Shaft	Steel	Bearing Load F	15000 N
	ambient Temperature	Rotational Speed N	30 1/min
	good heat conditions	R_a	0.3 µm

Calculation Constants and Application Factors		
Specific Load Limit p_{lim}	70 N/mm²	(Table 5, Page 13)
Application Factor a_T	1.0	(Fig. 18, Page 17)
Mating Surface Application Factor a_S	0.98	(Fig. 19, Page 17)
Bearing Size Factor a_B for ø 40	0.98	(Fig. 20, Page 18)
Application Factor for PM bush a_Q	1.8	(Fig. 16, Page 16)

Calculation	Ref	Value
Specific Load p [N/mm²]	(5.1.1), Page 13	$\bar{p} = \frac{F}{D_i \cdot B} = \frac{15000}{40 \cdot 30} = 12.5$
Sliding Speed U [m/s]	(5.2.1), Page 14	$U = \frac{D_i \cdot \pi \cdot N}{60 \cdot 10^3} = \frac{40 \cdot \pi \cdot 30}{60000} = 0.063$
High Load Factor a_E [-] (must be >0)	(5.8.1), Page 19	$a_E = \frac{\bar{p}_{lim}}{\bar{p}_{lim} - \bar{p}} = \frac{70}{70 - 12.5} = 1.22$
epU Factor [-]	(5.8.3), Page 19	$e\bar{p}U = \frac{a_E \cdot \bar{p}U}{a_B} = \frac{1.22 \cdot 12.5 \cdot 0.063}{0.98} = 0.98$
Life L_H [h] for $e\bar{p}U$<1	(5.8.3), Page 19	$L_H = \frac{3000}{e\bar{p}U} \cdot a_Q \cdot a_T \cdot a_S = \frac{3000}{0.98} \cdot 1.8 \cdot 1.0 \cdot 0.98 = 5400$
L_{RG} [h]	(5.8.3), Page 19	$L_{RG} = \frac{L_H}{2} = \frac{5400}{2} = 2700$

PM cylindrical Bush

Given:			
Load Details	Steady Load	Inside Diameter D_i	90 mm
	Direction: up	Length B	60 mm
Shaft	Steel	Bearing Load F	45000 N
	Temperature 80° C	Rotational Speed N	20 1/min
	good heat conditions	R_a	0.3 µm

Calculation Constants and Application Factors		
Specific Load Limit p_{lim} at 80 °C	46.7 N/mm²	(Table 5, Page 13)
Application Factor a_T	0.4	(Fig. 18, Page 17)
Mating Surface Application Factor a_S	0.98	(Fig. 19, Page 17)
Bearing Size Factor a_B for ø 40	0.70	(Fig. 20, Page 18)
Application Factor for PM bush a_Q	1.0	(Fig. 16, Page 16)

Calculation	Ref	Value
Specific Load p [N/mm²]	(5.1.1), Page 13	$\bar{p} = \frac{F}{D_i \cdot B} = \frac{45000}{90 \cdot 60} = 8.33$
Sliding Speed U [m/s]	(5.2.1), Page 14	$U = \frac{D_i \cdot \pi \cdot N}{60 \cdot 10^3} = \frac{90 \cdot \pi \cdot 20}{60000} = 0.094$
High Load Factor a_E [-] (must be >0)	(5.8.1), Page 19	$a_E = \frac{\bar{p}_{lim}}{\bar{p}_{lim} - \bar{p}} = \frac{46.7}{46.7 - 8.33} = 1.22$
epU Factor [-]	(5.8.3), Page 19	$e\bar{p}U = \frac{a_E \cdot \bar{p}U}{a_B} = \frac{1.22 \cdot 8.33 \cdot 0.094}{0.70} = 1.36$
Life L_H [h] for $e\bar{p}U$>1	(5.8.3), Page 19	$L_H = \frac{3000}{(e\bar{p}U)^{2.4}} \cdot a_Q \cdot a_T \cdot a_S = \frac{3000}{1.36^{2.4}} \cdot 1.0 \cdot 0.4 \cdot 0.98 = 562$
L_{RG} [h]	(5.8.3), Page 19	$L_{RG} = \frac{L_H}{2} = \frac{562}{2} = 281$

Thrust washer

Given:			
Load Details	Steady Load	Inside Diameter D_i	26 mm
	Direction: down	Outside Diameter D_o	44 mm
Shaft	Steel	Bearing Load F	10000 N
	ambient Temperature	Rotational Speed N	10 1/min
	good heat conditions	R_a	0.3 µm

Calculation Constants and Application Factors		
Specific Load Limit p_{lim}	70 N/mm²	(Table 5, Page 13)
Application Factor a_T	1.0	(Fig. 18, Page 17)
Mating Surface Application Factor a_S	0.98	(Fig. 19, Page 17)
Bearing Size Factor a_B for ø 35	0.90	(Fig. 20, Page 18)
Application Factor for Thrust washers a_Q	1.0	(Fig. 16, Page 16)

Calculation	Ref	Value
Specific Load p [N/mm²]	(5.1.2), Page 13	$\bar{p} = \frac{4 \cdot F}{\pi \cdot (D_o^2 - D_i^2)} = \frac{4 \cdot 10000}{\pi \cdot (44^2 - 26^2)} = 10.11$
Sliding Speed U [m/s]	(5.2.2), Page 14	$U = \frac{\frac{D_o + D_i}{2} \cdot \pi \cdot N}{60 \cdot 10^3} = \frac{\frac{44 + 26}{2} \cdot \pi \cdot 10}{60 \cdot 10^3} = 0.018$
High Load Factor a_E [-] (must be >0)	(5.8.1), Page 19	$a_E = \frac{\bar{p}_{lim}}{\bar{p}_{lim} - \bar{p}} = \frac{70}{70 - 10.11} = 1.169$
epU Factor [-]	(5.8.2), Page 19	$e\bar{p}U = \frac{a_E \cdot \bar{p}U}{a_B} = \frac{1.169 \cdot 10.11 \cdot 0.018}{0.90} = 0.236$
Life L_H [h] for $e\bar{p}U$<1	(5.8.3), Page 19	$L_H = \frac{3000}{e\bar{p}U} \cdot a_Q \cdot a_T \cdot a_S = \frac{3000}{0.236} \cdot 1.0 \cdot 1.0 \cdot 0.98 = 12460$
L_{RG} [h]	(5.8.3), Page 19	$L_{RG} = \frac{L_H}{2} = \frac{12460}{2} = 6230$

Slideways

Given:			
Load Details	Steady Load	Length L	50 mm
	Direction: down	Width W	20 mm
Mating Surface	Steel (R_a = 0.3 µm)	Bearing Load F	20000 N
	Temperature 80° C	Stroke	15 mm
	good heat conditions	Frequency	10 1/min

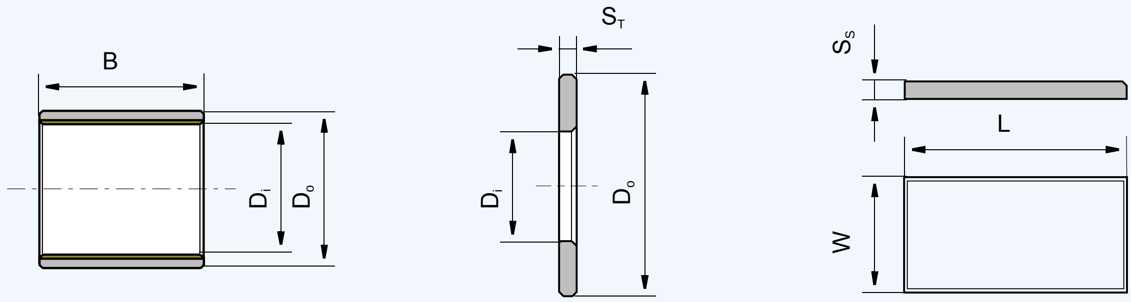
Calculation Constants and Application Factors		
Specific Load Limit p_{lim} at 80 °C	93 N/mm²	(Table 5, Page 13)
Application Factor a_T	0.4	(Fig. 18, Page 17)
Mating Surface Application Factor a_S	0.98	(Fig. 19, Page 17)
Bearing Size Factor a_B	1.0	(Fig. 20, Page 18)
Application Factor for Slideways a_Q	1.0	(Fig. 16, Page 16)

Calculation	Ref	Value
Specific Load p [N/mm²]	(5.1.3), Page 13	$\bar{p} = \frac{F}{L \cdot W} = \frac{20000}{50 \cdot 20} = 20$
Sliding Speed U [m/s]		$U = \frac{15 \cdot 2 \cdot 10}{60 \cdot 10^3} = 0.005$
High Load Factor a_E [-] (must be >0)	(5.8.1), Page 19	$a_E = \frac{\bar{p}_{lim}}{\bar{p}_{lim} - \bar{p}} = \frac{93}{93 - 20} = 1.27$
epU Factor [-]	(5.8.2), Page 19	$e\bar{p}U = \frac{a_E \cdot \bar{p}U}{a_B} = \frac{1.27 \cdot 20 \cdot 0.005}{1.0} = 0.127$
Life L_H [h] for $e\bar{p}U$<1	(5.8.3), Page 19	$L_H = \frac{3000}{e\bar{p}U} \cdot a_Q \cdot a_T \cdot a_S = \frac{3000}{0.127} \cdot 1.0 \cdot 0.4 \cdot 0.98 = 9260$
L_{RG} [h]	(5.8.3), Page 19	$L_{RG} = \frac{L_H}{2} = \frac{9260}{2} = 4630$

6 Data Sheet

Application:

6.1 Data for bearing design calculations



☐ Cylindrical Bush ☐ Thrust Washer ☐ Slideplate ☐ Special (Sketch)

☐ Rotational movement ☐ Steady load ☐ Rotating load ☐ Oscillating movement ☐ Linear movement

☐ Existing Design ☐ New Design

Quantity

Dimensions in mm

Inside Diameter	D_i	
Outside Diameter	D_o	
Width	B	
Length of slideplate	L	
Width of slideplate	W	
Thickness of slideplate	S_S	

Load

Radial load	F [N]	
or specific load	$\bar{p}$ [N/mm^2]	
Axial load	F [N]	
or specific load	$\bar{p}$ [N/mm^2]	

Movement

Rotational speed	N [1/min]	
Speed	U [m/s]	
Length of Stroke	L_S [mm]	
Frequency of Stroke	[1/min]	
Oscillating cycle	φ [°]	
Oscillating frequency	N_{osz} [1/min]	

Service hours per day

Continuous operation	
Intermittent operation	
Operating time	
Days per year	

Fits and Tolerances

Shaft	D_J	
Bearing Housing	D_H	

Operating Environment

Ambient temperature	T_{amb} [°]	

Housing with good heat transfer properties ☐

Light pressing or insulated housing with poor heat transfer properties ☐

Non metal housing with poor heat transfer properties ☐

Alternate operation in water and dry ☐

Mating surface

Material		
Hardness	HB/HRC	
Surface finish	R_a [µm]	

Lubrication

Dry ☐

Continuous lubrication ☐

Process fluid lubrication ☐

Initial lubrication only ☐

Hydrodynamic conditions ☐

Process Fluid		
Lubricant		
Dynamic viscosity	η	

Service life

Required service life	L_H [h]	

Customer Data
Company: City:
Street: Post Code:

Project:
Name:
Tel.:

Date:
Signature:
Fax:

7 Bearing Assembly

7.1 Dimensions and Tolerances

For optimum performance it is essential that the correct running clearance is used and that both the diameter of the shaft and the bore of the housing are finished to the limits given in the tables.

If the bearing housing is unusually flexible the bush will not close in by the calculated amount and the running clearance will be more than the optimum. In these circumstances the housing should be bored slightly undersize or the journal diameter increased, the correct size being determined by experiment.

7.2 Tolerances for minimum clearance

Grease lubrication

The minimum clearance required for satisfactory performance of DX depends upon the $\bar{p}U$ factor, the sliding speed and the environmental temperature, any one or combination of which may reduce the diametral clearance in operation due to inward thermal expansion of the DX polymer lining. It is therefore necessary to compensate for this.

Fig. 21 shows the minimum diametral clearance plotted stepped against journal diameter at an ambient 20°C. Where the stepped lines show a change of clearance for a given journal diameter, the lower value is used.

The superimposed straight lines indicate the minimum permissible diametral clearance for various values of $\bar{p}Uu$ (Fig. 21), where $\bar{p}U$ is calculated as in 5.3 on Page 15, and u is a sliding speed factor for speeds in excess of 0.5 m/s given in Fig. 22.

If the clearance indicated for a $\bar{p}Uu$ factor lies below the stepped lines the recommended standard shaft may be used. If above, the shaft size must be reduced to obtain the clearance indicated on the vertical axis of the relevant figure.

Under slow speed and high load conditions it may be possible to achieve satisfactory performance with diametral clearances less than those indicated. But adequate prototype testing is recommended in such cases.

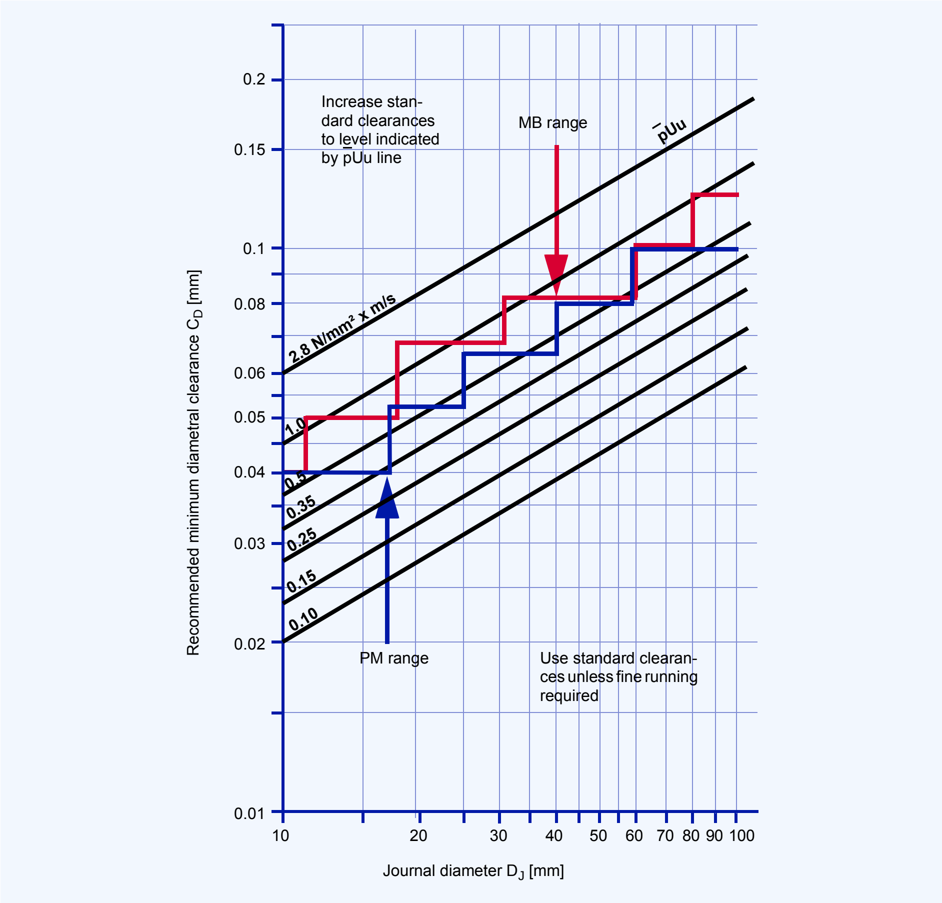


Fig. 21: Minimum clearance for PM prefinished and MB machinable metric range machined to H7 bore

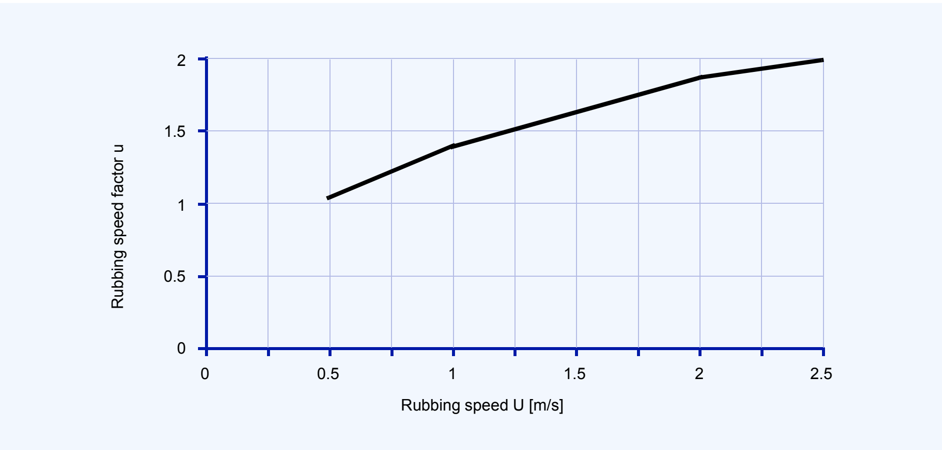


Fig. 22: Rubbing speed factor u

Fluid Lubrication

The minimum clearance required for journal bearings operating under hydrodynamic or mixed film conditions for a range of shaft rotational speeds and diameters is shown in Fig. 23 It is recommended that the bearing performance under minimum clearance conditions be confirmed by testing if possible.

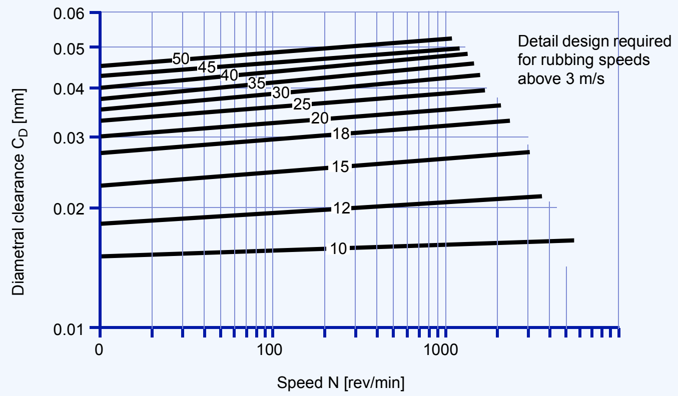


Fig. 23: DX minimum clearances - bush diameters D_i 10-50 mm

Allowance for Thermal Expansion

For operation in high temperature environments the clearance should be increased by the amounts indicated by Fig. 24 to compensate for the inward thermal expansion of the bearing lining.

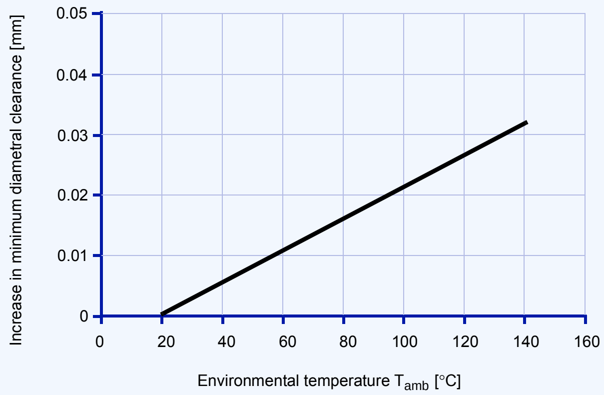


Fig. 24: Recommended increase in diametral clearance

If the housing is non-ferrous then the bore should be reduced by the amounts given in Table 6, in order to give an increased interference fit to the bush, with a similar reduction in the journal diameter additional to that indicated by Fig. 24.

Housing material	Reduction in housing diameter per 100°C rise	Reduction in shaft diameter per 100°C rise
Aluminium alloys	0.1%	0.1% + values from Fig. 24
Copper base alloys	0.05%	0.05% + values from Fig. 24
Steel and cast iron	Nil	values from Fig. 24
Zinc base alloys	0.15%	0.15% + values from Fig. 24

Table 6: Allowance for high temperature

7.3 Counterface Design

DX bearings may be used with all conventional mating surface materials. Hardening of steel journals is not required unless abrasive dirt is present or if the projected bearing life is in excess of 2000 hours, in which cases a minimum shaft hardness of 350HB is recommended.

A ground surface finish of better than 0.4µm R_a is recommended. The final direction of machining of the mating surface should preferably be the same as the direction of motion relative to the bearing in service.

DX is normally used in conjunction with ferrous journals and thrust faces, but in damp or corrosive surroundings stainless steel or hard chromium plated mild steel, alternatively WH shaft sleeves (Standard programm available) are recommended. When plated mating surfaces are specified the plating should possess adequate strength and adhesion, particularly if the bearing is to operate with high fluctuating loads.

The shaft or thrust collar used in conjunction with the DX bush or thrust washer must extend beyond the bearing surface in order to avoid cutting into it. The mating surface must also be free from grooves or flats, the end of the shaft should be given a lead-in chamfer and all sharp edges or projections which may damage the soft polymer lining of the DX must be removed.

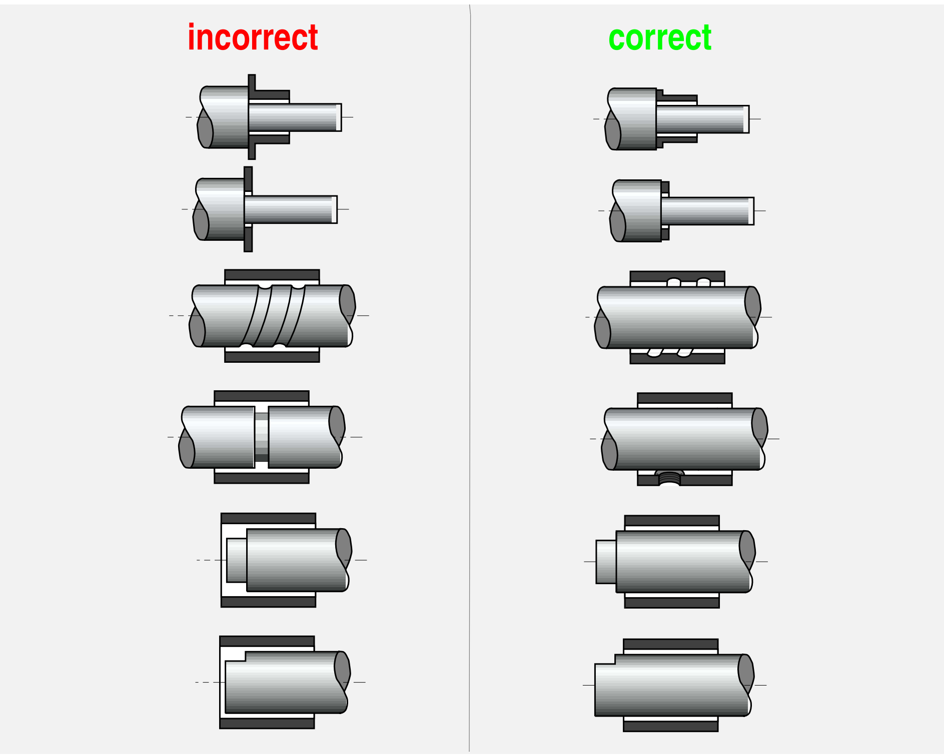


Fig. 25: Counterface design

7.4 Installation

Important Note

Care must be taken to ensure that the DX lining material is not damaged during the installation.

Fitting of Bushes

The bush is inserted into its housing with the aid of a stepped mandrel, preferably made from case hardened mild steel, as shown in Fig. 26. The following should be noted to avoid damage to the bearing:

- Housing diameter is as recommended
- 15-30° lead-in chamfer on housing
- edges of lead-in chamfer are deburred
- The bush must be square to the housing
- Light smear of oil on bush OD

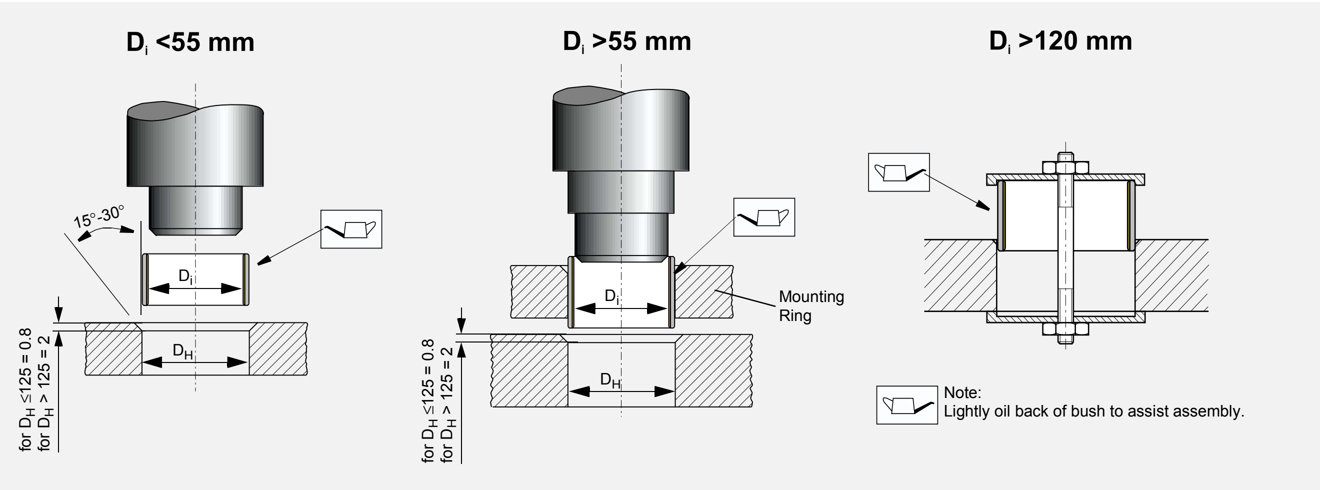


Fig. 26: Fitting of bushes

Insertion Forces

Fig. 27 gives an indication of the maximum insertion force required to correctly install standard DX bushes.

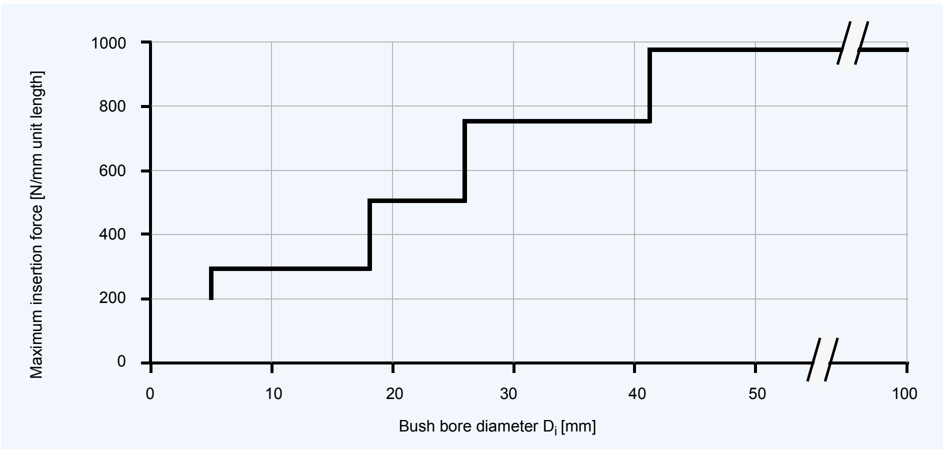


Fig. 27: Maximum insertion force F_i

Alignment

Accurate alignment is an important consideration for all bearing assemblies. With DX bearings misalignment over the length of a bush (or pair of bushes), or over the diameter of a thrust washer should not exceed 0.020 mm as illustrated in Fig. 28.

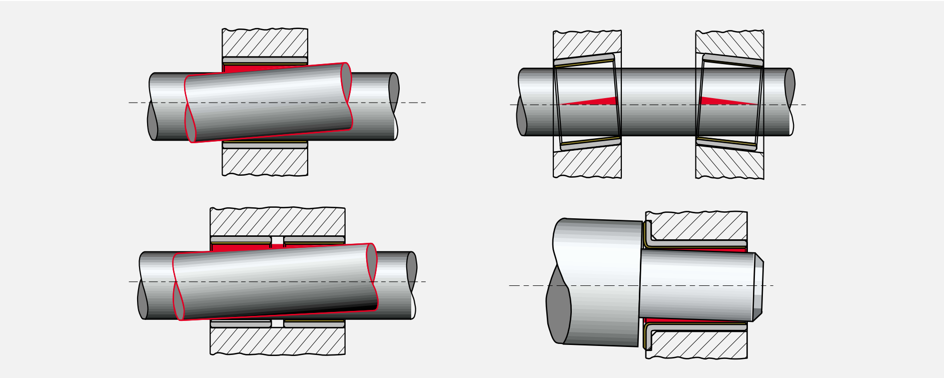


Fig. 28: Alignment

Sealing

While DX can tolerate the ingress of some contaminant materials into the bearing without loss of performance, where there is the possibility of highly abrasive material entering the bearing, a suitable sealing arrangement, as illustrated in Fig. 29 should be provided.

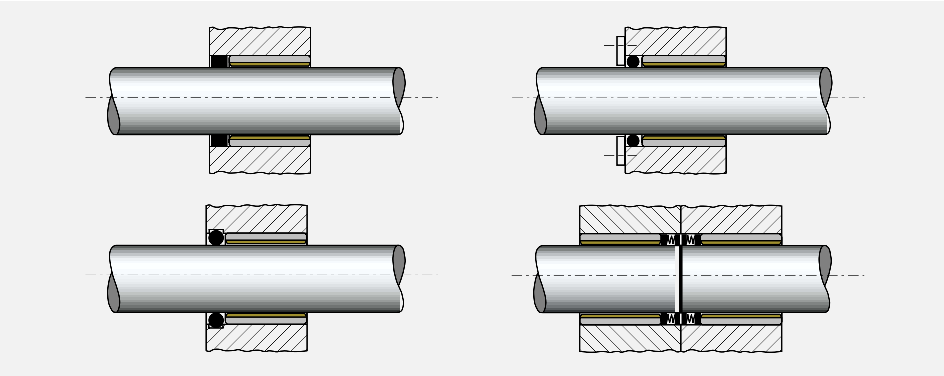


Fig. 29: Recommended sealing arrangements

Axial Location

Where axial location is necessary, it is generally advisable to fit DX thrust washers in conjunction with DX bushes, even when the axial loads are low. Experience has shown that fretting debris from unsatisfactory locating surfaces can enter an adjacent DX bush and adversely affect the bearing life and performance.

Fitting of Thrust Washers

DX thrust washers should be located on the outside diameter in a recess as shown in Fig. 30. The inside diameter must be clear of the shaft in order to prevent contact with the steel backing of the DX material. The recess diameter should be 0.125 mm larger than the washer diameter and the depth as given in the product tables.

If there is no recess for the thrust washer one of the following methods of fixing may be used:

- two dowel pins
- two screws
- adhesive

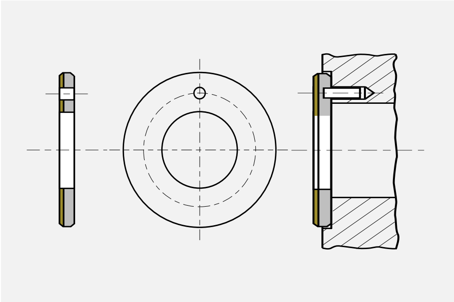


Fig. 30: Installation of Thrust-Washer

Important Note

- Dowel pins should be recessed 0.25 mm below the bearing surface
- Screws should be countersunk 0.25 mm below the bearing surface
- DX must not be heated above 130 °C
- Contact adhesive manufacturers for guidance on the selection of suitable adhesives
- Protect the bearing surface to prevent contact with adhesive
- Ensure the washer ID does not touch the shaft after assembly
- Ensure that the washer is mounted with the steel backing to the housing

Slideways

DX strip material for use as slideway bearings should be installed using one of the following methods:

- countersunk screws
- adhesives
- mechanical location

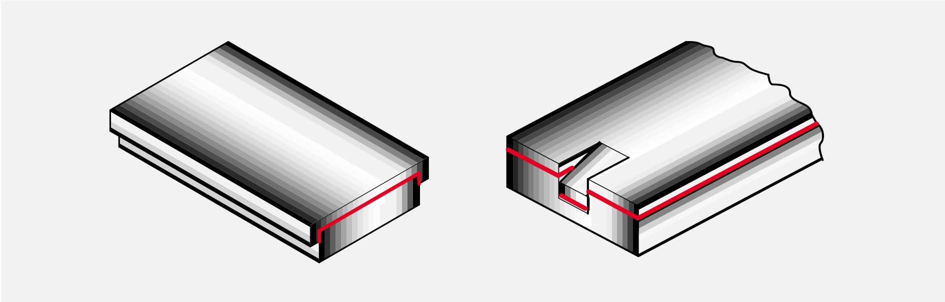


Fig. 31: Mechanical location of DX slideways

8 Machining

8.1 Machining Practice

The acetal copolymer lining of DX has good machining characteristics and can be treated as a free cutting brass in most respects. The indents in the bearing surface may lead to the formation of burrs or whiskers due to the resilience of the lining material, but this can be avoided by using machining methods which remove the lining as a ribbon, rather than a narrow thread.

When machining DX it is recommended that not more than 0.125 mm is removed from the lining thickness in order to ensure that the lubricant capacity of the indents remaining after machining is not significantly reduced.

Boring, reaming and broaching are all suitable machining methods for use with DX. The recommended tool material is high speed steel or tungsten carbide.

8.2 Boring

Fig. 32 illustrates a recommended boring tool which should be mounted with its axis at right angles to the direction of feed.

The essential characteristic required in the boring tool is a tip radius greater than 1.5 mm, which combined with a side rake of 30° will produce the ribbon effect required.

Cutting speeds should be high, the optimum between 2.0 and 4.5 m/s. The feed should be low, in the range 0.05/0.025 mm for cuts of 0.125 mm, the lower feeds being used with the higher cutting speeds.

Satisfactory finishes can usually be obtained machining dry and an air blast may facilitate swarfe removal. The use of coolant is not detrimental.

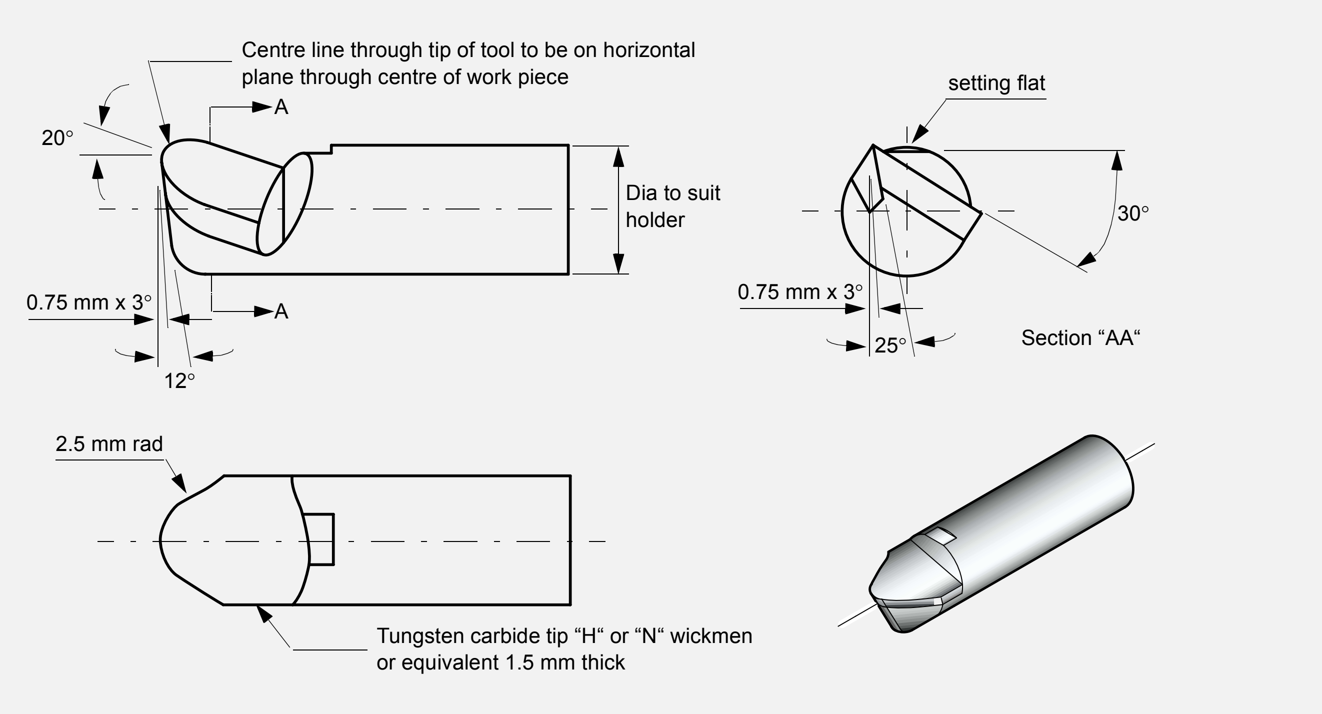


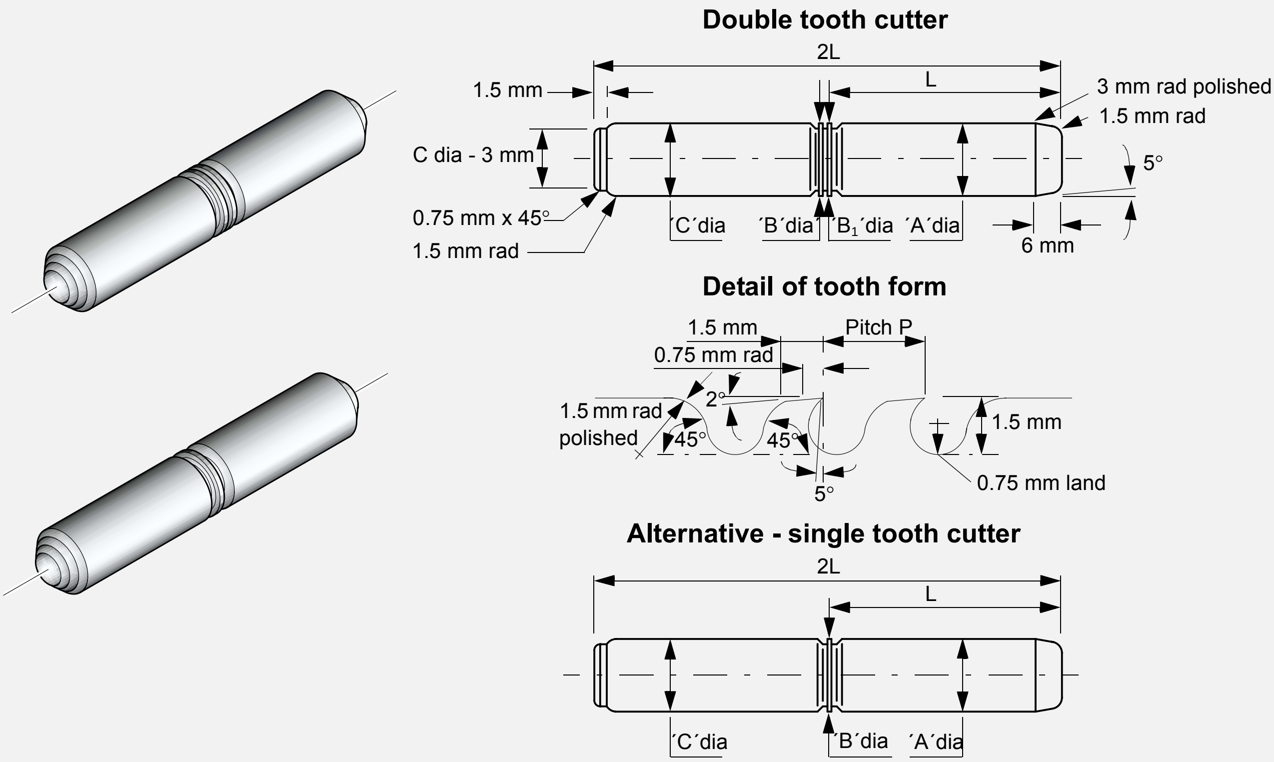
Fig. 32: Boring tool for DX

8.3 Reaming

DX can be reamed satisfactorily by hand with a straight-fluted expanding reamer. For best results the reamer should be sharp, the cut 0.025-0.050 mm and the feed slow. Where hand reaming is not desired machining speeds of about 0.05 m/s are recommended with the cuts and feeds as for boring.

8.4 Broaching

Fig. 33 shows broaches suitable for finishing bushes up to 65 mm diameter. The broach should be used dry, at a speed of 0.1-0.5 m/s.



Bush Width		Pitch
B		P
Over	To	
10	13	3
13	20	4
20	30	5
30	50	5.5
50	70	6
70	95	7
95	130	8

Diameter		
´A´	Min. ass. bore	+0.013 +0
´B´	Nominal bore	+0.038 +0.025
´C´	Nominal bore	+0.015 +0.005
Min. ass. bore = $D_{o\ min}$ - 2 x $S_{3\ max}$		
Nominal bore = min. finished bore		
´B_1´*	Nominal bore	-0.065 -0.076

	Min. length of Pilot Guide L_{min}
Single bush	B + 6
2 or more bushes in line	B + 6 + bush spacing

* First tooth of double tooth cutter

Fig. 33: Suitable broaches for DX

Use the single tooth version where the bush is less than 25 mm long, and the double tooth broach for longer bushes or for two or more bushes together.

If it is necessary to make up a special form of broach the following points should be noted:

- Adequate provision should be made for locating the bush by providing a pilot to suit the bore of the bush when pressed home. A rear support shoulder should locate in the broached bore of the bush after cutting. Alternatively, special guides may be provided external to the workpiece.
- If two bushes are to be broached in line,

then the pilot guide and rear support should be longer than the distance between the two bushes.

- For large bushes it may be necessary to provide axial relief along the length of the pilot guide and rear support, in order to reduce the broaching forces.
- Unless a guided broach is used, the tool will follow the initial bore alignment of the bush, broaching cannot improve concentricity and parallelism unless external guides are used.

In general owing to the variation in wall thickness of large diameter bushes, broaching is not suitable for finishing bores of more than 60 mm diameter unless external guides are used.

8.5 Vibrobroaching

This technique may also be used. A single cutter is propelled with progressive reciprocating motion with a vibration frequency of typically 50 Hz. The cutter should have a primary rake of 1.5° for 0.5 mm.

A cut of 0.25 mm on diameter may be made at an average cutting speed of 0.15 m/s to give a surface finish of better than 0.8 µm R_a, which is acceptable.

8.6 Modification of components

The modification of DX bearing components requires no special procedures. In general it is more satisfactory to perform machining or drilling operations from the polymer lining side in order to avoid burrs. When cutting is done from the steel side, the minimum cutting pressure should be used and care taken to ensure that any steel or bronze particles protruding into the remaining bearing material, and all burrs, are removed.

8.7 Drilling Oil Holes

Bushes should be adequately supported during the drilling operation to ensure that no distortion is caused by the drilling pressure.

8.8 Cutting Strip Material

DX strip material may be cut to size by any one of the following methods. Care must be taken to protect the bearing surface from damage and to ensure that no deformation of the strip occurs.

- Using side and face cutter, or slitting saw, with the strip held flat and securely on a horizontal milling machine.
- Cropping
- Guillotine (For widths less than 90 mm only)
- Water-jet cutting, Laser cutting

9 Electroplating

DX Components

To provide corrosion protection the mild steel backing of DX may be electroplated with most of the conventional electroplating metals including the following:

- zinc ISO 2081-2
- cadmium ISO 2081-2
- nickel ISO 1456-8
- hard chromium ISO 1456-8

For the harder materials if the specified plating thickness exceeds approximately 5 µm then the housing diameter should be increased by twice the plating thickness in order to maintain the correct assembled bearing bore size.

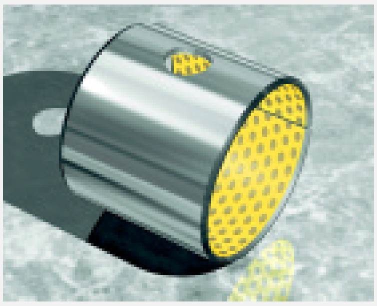
Where electrolytic attack is possible tests should be conducted to ensure that all the materials in the bearing environment are mutually compatible.

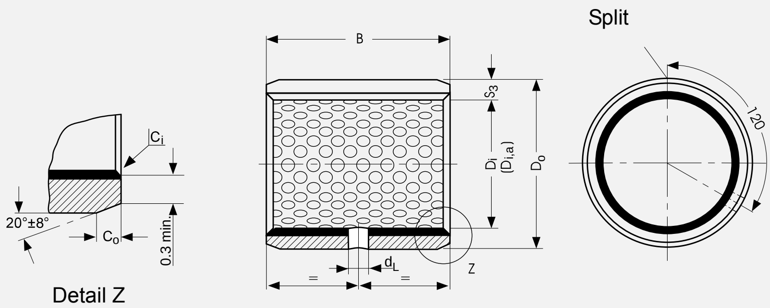
Mating Surfaces

DX can be used against hard chrome plated materials and care should be taken to ensure that the recommended shaft sizes and surface finish are achieved after the plating process.

10 Standard Products

10.1 PM-DX cylindrical bushes





Dimensions and tolerances follow ISO 3547 and GSP-Specifications

Part No.	Nominal size		Width B	Wall thickness S_3	Shaft-ø D_J [h8]	Housing-ø D_H [H7]	Bush i-ø $D_{i,a}$ assembled in H7 housing	Clearance C_D	Oil hole -ø d_L
	D_i	D_o	max. min.	max. min.	max. min.	max. min.	max. min.	max. min.	
PM 0808 DX			8.25 7.75						
PM 0810 DX	8	10	10.25 9.75		8.000 7.978	10.015 10.000	8.105 8.040	0.127 0.040	No hole
PM 0812 DX			12.25 11.75						
PM 1010 DX			10.25 9.75						3
PM 1012 DX	10	12	12.25 11.75		10.000 9.978	12.018 12.000	10.108 10.040	0.130 0.040	
PM 1015 DX			15.25 14.75						4
PM 1020 DX			20.25 19.75						
PM 1210 DX			10.25 9.75						3
PM 1212 DX			12.25 11.75						
PM 1215 DX	12	14	15.25 14.75	0.980 0.955	12.000 11.973	14.018 14.000	12.108 12.040		
PM 1220 DX			20.25 19.75						
PM 1225 DX			25.25 24.75						4
PM 1415 DX			15.25 14.75						
PM 1420 DX	14	16	20.25 19.75		14.000 13.973	16.018 16.000	14.108 14.040	0.135 0.040	
PM 1425 DX			25.25 24.75						
PM 1510 DX			10.25 9.75						3
PM 1512 DX	15	17	12.25 11.75		15.000 14.973	17.018 17.000	15.108 15.040		
PM 1515 DX			15.25 14.75						4
PM 1525 DX			25.25 24.75						

All dimensions in mm

ID and OD chamfers

S_3	C_o	C_i	S_3	C_o	C_i
1	0.6 ± 0.4	–0.1 to –0.5	2	1.2 ± 0.4	–0.1 to –0.7
1.5	0.6 ± 0.4	–0.1 to –0.7	2.5	1.6 ± 0.8	–0.2 to –1.0

10 Standard Products

Part No.	Nominal size		Width B	Wall thickness S_3	Shaft-ø D_J [h8]	Housing-ø D_H [H7]	Bush i-ø $D_{i,a}$ assembled in H7 housing	Clearance C_D	Oil hole -ø d_L
	D_i	D_o	max. min.	max. min.	max. min.	max. min.	max. min.	max. min.	
PM 1615 DX	16	18	15.25 14.75	0.980 0.955	16.000 15.973	18.018 18.000	16.108 16.040	0.135 0.040	4
PM 1620 DX			20.25 19.75						
PM 1625 DX			25.25 24.75						
PM 1815 DX	18	20	15.25 14.75		18.000 17.973	20.021 20.000	18.111 18.040	0.138 0.040	
PM 1820 DX			20.25 19.75						
PM 1825 DX			25.25 24.75						
PM 2010 DX	20	23	10.25 9.75	1.475 1.445	20.000 19.967	23.021 23.000	20.131 20.050	0.164 0.050	
PM 2015 DX			15.25 14.75						
PM 2020 DX			20.25 19.75						
PM 2025 DX			25.25 24.75						
PM 2030 DX			30.25 29.75						
PM 2215 DX	22	25	15.25 14.75		22.000 21.967	25.021 25.000	22.131 22.050		6
PM 2220 DX			20.25 19.75						
PM 2225 DX			25.25 24.75						
PM 2230 DX			30.25 29.75						
PM 2415 DX	24	27	15.25 14.75		24.000 23.967	27.021 27.000	24.131 24.050		
PM 2420 DX			20.25 19.75						
PM 2425 DX			25.25 24.75						
PM 2430 DX			30.25 29.75						
PM 2515 DX	25	28	15.25 14.75		25.000 24.967	28.021 28.000	25.131 25.050		
PM 2520 DX			20.25 19.75						
PM 2525 DX			25.25 24.75						
PM 2530 DX			30.25 29.75						
PM 2830 DX	28	31	30.25 29.75		28.000 27.967	31.025 31.000	28.135 28.050	0.168 0.050	
PM 2820 DX		32	20.25 19.75	1.970 1.935		32.025 32.000	28.155 28.060	0.188 0.060	
PM 2825 DX			25.25 24.75						
PM 2830 DX			30.25 29.75						
PM 3020 Dx	30	34	20.25 19.75		30.000 29.967	34.025 34.000	30.155 30.060		
PM 3030 DX			30.25 29.75						
PM 3040 DX			40.25 39.75						

Part No.	Nominal size D_i	Nominal size D_o	Width B max. min.	Wall thickness S_3 max. min.	Shaft-ø D_J [h8] max. min.	Housing-ø D_H [H7] max. min.	Bush i-ø $D_{i,a}$ assembled in H7 housing max. min.	Clearance C_D max. min.	Oil hole -ø d_L
PM 3220 DX	32	36	20.25 19.75	1.970 1.935	32.000 31.961	36.025 36.000	32.155 32.060	0.194 0.060	6
PM 3230 DX			30.25 29.75						
PM 3235 DX			35.25 34.75						
PM 3240 DX			40.25 39.75						
PM 3520 DX	35	39	20.25 19.75		35.000 34.961	39.025 39.000	35.155 35.060		
PM 3530 DX			30.25 29.75						
PM 3535 DX			35.25 34.75						
PM 3550 DX			50.25 49.75						
PM 3635 DX	36	40	35.25 34.75		36.000 35.961	40.025 40.000	36.155 36.060		
PM 3720 DX	37	41	20.25 19.75		37.000 36.961	41.025 41.000	37.155 37.060		
PM 4020 DX	40	44	20.25 19.75		40.000 39.961	44.025 44.000	40.155 40.060		8
PM 4030 DX			30.25 29.75						
PM 4040 DX			40.25 39.75						
PM 4050 DX			50.25 49.75						
PM 4520 DX	45	50	20.25 19.75	2.460 2.415	45.000 44.961	50.025 50.000	45.195 45.080	0.234 0.080	
PM 4530 DX			30.25 29.75						
PM 4540 DX			40.25 39.75						
PM 4545 DX			45.25 44.75						
PM 4550 DX			50.25 49.75						
PM 5040 DX	50	55	40.25 39.75		50.000 49.961	55.030 55.000	50.200 50.080	0.239 0.080	
PM 5050 DX			50.25 49.75						
PM 5060 DX			60.25 59.75						
PM 5520 DX	55	60	20.25 19.75		55.000 54.954	60.030 60.000	55.200 55.080	0.246 0.080	
PM 5525 DX			25.25 24.75						
PM 5530 DX			30.25 29.75						
PM 5540 DX			40.25 39.75						
PM 5550 DX			50.25 49.75						
PM 5560 DX			60.25 59.75						
PM 6030 DX	60	65	30.25 29.75		60.000 59.954	65.030 65.000	60.200 60.080		
PM 6040 DX			40.25 39.75						
PM 6060 DX			60.25 59.75						
PM 6070 DX			70.25 69.75						

10 Standard Products

Part No.	Nominal size		Width B	Wall thickness S_3	Shaft-ø D_J [h8]	Housing-ø D_H [H7]	Bush i-ø $D_{i,a}$ assembled in H7 housing	Clearance C_D	Oil hole -ø d_L
	D_i	D_o	max. min.	max. min.	max. min.	max. min.	max. min.	max. min.	
PM 6540 DX	65	70	40.25 39.75	2.450 2.384	65.000 64.954	70.030 70.000	65.262 65.100	0.308 0.100	8
PM 6550 DX			50.25 49.75						
PM 6560 DX			60.25 59.75						
PM 6570 DX			70.25 69.75						
PM 7040 DX	70	75	40.25 39.75		70.000 69.954	75.030 75.000	70.262 70.100		
PM 7050 DX			50.25 49.75						
PM 7065 DX			65.25 64.75						
PM 7070 DX			70.25 69.75						
PM 7080 DX			80.25 79.75						
PM 7540 DX	75	80	40.25 39.75		75.000 74.954	80.030 80.000	75.262 75.100		9.5
PM 7560 DX			60.25 59.75						
PM 7580 DX			80.25 79.75						
PM 8040 DX	80	85	40.50 39.50		80.000 79.954	85.035 85.000	80.267 80.100	0.313 0.100	
PM 8060 DX			60.50 59.50						
PM 8080 DX			80.50 79.50						
PM 80100 DX			100.50 99.50						
PM 8530 DX	85	90	30.50 29.50		85.000 84.946	90.035 90.000	85.267 85.100	0.321 0.100	
PM 8540 DX			40.50 39.50						
PM 8560 DX			60.50 59.50						
PM 8580 DX			80.50 79.50						
PM 85100 DX			100.50 99.50						
PM 9040 DX	90	95	40.50 39.50		90.000 89.946	95.035 95.000	90.267 90.100		
PM 9060 DX			60.50 59.50						
PM 9080 DX			80.50 79.50						
PM 9090 DX			90.50 89.50						
PM 90100 DX			100.50 99.50						
PM 9560 DX	95	100	60.50 59.50		95.000 94.946	100.035 100.000	95.267 95.100		
PM 95100 DX			100.50 99.50						

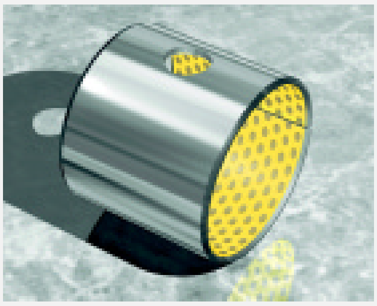
Part No.	Nominal size D_i	Nominal size D_o	Width B max. min.	Wall thickness S_3 max. min.	Shaft-ø D_J [h8] max. min.	Housing-ø D_H [H7] max. min.	Bush i-ø $D_{i,a}$ assembled in H7 housing max. min.	Clearance C_D max. min.	Oil hole -ø d_L
PM 10050 DX	100	105	50.50 49.50	2.450 2.384	100.000 99.946	105.035 105.000	100.267 100.100	0.321 0.100	9.5
PM 10060 DX			60.50 59.50						
PM 10080 DX			80.50 79.50						
PM 10095 DX			95.50 94.50						
PM 100115DX			115.50 114.50						
PM 10560 DX	105	110	60.50 59.50		105.000 104.946	110.035 110.000	105.267 105.100		
PM 105110 DX			110.50 109.50						
PM 105115 DX			115.50 114.50						
PM 11060 DX	110	115	60.50 59.50		110.000 109.946	115.035 115.000	110.267 105.100		
PM 110110 DX			110.50 109.50						
PM 110115 DX			115.50 114.50						
PM 11550 DX	115	120	50.50 49.50		115.000 114.946	120.035 120.000	115.267 115.100		
PM 11570 DX			70.50 69.95						
PM 12060 DX	120	125	60.50 59.50		120.000 119.946	125.040 125.000	120.272 120.100	0.326 0.100	
PM 120100 DX			100.50 99.50						
PM 120110 DX			110.50 109.50						
PM 12560 DX	125	130	60.50 59.50		125.000 124.937	130.040 130.000	125.272 125.100	0.335 0.100	
PM 125100 DX			100.50 99.50						
PM 125110 DX			110.50 109.50						
PM 13050 DX	130	135	50.50 49.50	2.435 2.380	130.000 129.937	135.040 135.000	130.280 130.130	0.343 0.130	-
PM 13060 DX			60.50 59.50						
PM 13080 DX			80.50 79.50						
PM 130100 DX			100.50 99.50						
PM 13560 DX	135	140	60.50 59.50		135.000 134.937	140.040 140.000	135.280 135.130		
PM 13580 DX			80.50 79.50						
PM 14050 DX	140	145	50.50 49.50		140.000 139.937	145.040 145.000	140.280 140.130		
PM 14060 DX			60.50 59.50						
PM 14080 DX			80.50 79.50						
PM 140100 DX			100.50 99.50						
PM 15050 DX	150	155	50.50 49.50		150.000 149.937	155.040 155.000	150.280 150.130		
PM 15060 DX			60.50 59.50						
PM 15080 DX			80.50 79.50						
PM 150100 DX			100.50 99.50						

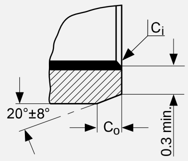
10 Standard Products

Part No.	Nominal size		Width B	Wall thickness S_3	Shaft-ø D_J [h8]	Housing-ø D_H [H7]	Bush i-ø $D_{i,a}$ assembled in H7 housing	Clearance C_D	Oil hole -ø d_L
	D_i	D_o	max. min.	max. min.	max. min.	max. min.	max. min.	max. min.	
PM 16050 DX	160	165	50.50 49.50	2.435 2.380	160.000 159.937	165.040 165.000	160.280 160.130	0.343 0.130	-
PM 16060 DX			60.50 59.50						
PM16080 DX			80.50 79.50						
PM 160100 DX			100.50 99.50						
PM 17050 DX	170	175	50.50 49.50		170.000 169.937	175.040 175.000	170.280 170.130		
PM 17060 DX			60.50 59.50						
PM 17080 DX			80.50 79.50						
PM 170100 DX			100.50 99.50						
PM 18050 DX	180	185	50.50 49.50		180.000 179.937	185.046 185.000	180.286 180.130	0.349 0.130	
PM 18060 DX			60.50 59.50						
PM 18080 DX			80.50 79.50						
PM 180100 DX			100.50 99.50						
PM 19050 DX	190	195	50.50 49.50		190.000 189.928	195.046 195.000	190.286 190.130	0.358 0.130	
PM 19060 DX			60.50 59.50						
PM 19080 DX			80.50 79.50						
PM 190100 DX			100.50 99.50						
PM 190120 DX			120.50 19.50						
PM 20050 DX	200	205	50.50 49.50		200.000 199.928	205.046 205.000	200.286 200.130		
PM 20060 DX			60.50 59.50						
PM 20080 DX			80.50 79.50						
PM 200100 DX			100.50 99.50						
PM 200120 DX			120.50 119.50						
PM 22050 DX	220	225	50.50 49.50		220.000 219.928	225.046 225.000	220.286 220.130		
PM 22060 DX			60.50 59.50						
PM 22080 DX			80.50 79.50						
PM 220100 DX			100.50 99.50						
PM 220120 DX			120.50 119.50						
PM 24050 DX	240	245	50.50 49.50		240.000 239.928	245.046 245.000	240.286 240.130		
PM 24060 DX			60.50 59.50						
PM 24080 DX			80.50 79.50						
PM 240100 DX			100.50 99.50						
PM 240120 DX			120.50 119.50						

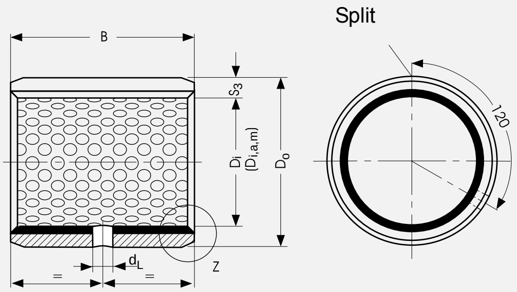
Part No.	Nominal size		Width B	Wall thickness S_3	Shaft-ø D_J [h8]	Housing-ø D_H [H7]	Bush i-ø $D_{i,a}$ assembled in H7 housing	Clearance C_D	Oil hole -ø d_L
	D_i	D_o	max. min.	max. min.	max. min.	max. min.	max. min.	max. min.	
PM 25050 DX	250	255	50.50 49.50	2.435 2.380	250.000 249.928	255.052 255.000	250.292 250.130	0.364 0.130	-
PM 25060 DX			60.50 59.50						
PM 25080 DX			80.50 79.50						
PM 250100 DX			100.50 99.50						
PM 250120 DX			120.50 119.50						
PM 26050 DX	260	265	50.50 49.50		260.000 259.919	265.052 265.000	260.292 260.130	0.373 0.130	
PM 26060 DX			60.50 59.50						
PM 26080 DX			80.50 79.50						
PM 260100 DX			100.50 99.50						
PM 260120 DX			120.50 119.50						
PM 28050 DX	280	285	50.50 49.50		280.000 279.919	285.052 285.000	280.292 280.130		
PM 28060 DX			60.50 59.50						
PM 28080 DX			80.50 79.50						
PM 280100 DX			100.50 99.50						
PM 280120 DX			120.50 119.50						
PM 30050 DX	300	305	50.50 49.50		300.000 299.919	305.052 305.000	300.292 300.130		
PM 30060 DX			60.50 59.50						
PM 30080 DX			80.50 79.50						
PM 300100 DX			100.50 99.50						
PM 300120 DX			120.50 119.50						

10.2 MB-DX cylindrical bushes





Detail Z



Dimensions and tolerances follow ISO 3547 and GSP-Specifications

Part No.	Nominal size		Width B	Wall thickness S_3	Shaft-ø D_J [d8]	Housing-ø D_H [H7]	Bush i-ø $D_{i,a,m}$ machined to [H7]	Clearance C_D	Oil hole -ø d_L
	D_i	D_o	max. min.	max. min.	max. min.	max. min.	max. min.	max. min.	
MB 0808 DX	8	10	8.25 7.75	1.108 1.082	7.960 7.938	10.015 10.000	8.015 8.000	0.077 0.040	No hole
MB 0810 DX			10.25 9.75						
MB 0812 DX			12.25 11.75						
MB 1010 DX	10	12	10.25 9.75		9.950 9.923	12.018 12.000	10.018 10.000	0.080 0.040	3
MB 1012 DX			12.25 11.75						4
MB 1015 DX			15.25 14.75						
MB 1020 DX			20.25 19.75						
MB 1210 DX	12	14	10.25 9.75		11.950 11.923	14.018 14.000	12.018 12.000	0.095 0.050	3
MB 1212 DX			12.25 11.75						4
MB 1215 DX			15.25 14.75						
MB 1220 DX			20.25 19.75						
MB 1225 DX			25.25 24.75						
MB 1415 DX	14	16	15.25 14.75		13.950 13.923	16.018 16.000	14.018 14.000		
MB 1420 DX			20.25 19.75						
MB 1425 DX			25.25 24.75						
MB 1510 DX	15	17	10.25 9.75		14.950 14.923	17.018 17.000	15.018 15.000		3
MB 1512 DX			12.25 11.75						4
MB 1515 DX			15.25 14.75						
MB 1525 DX			25.25 24.75						

ID and OD chamfers

S_3	C_o	C_i	S_3	C_o	C_i
1	0.6 ± 0.4	–0.1 to –0.5	2	1.2 ± 0.4	–0.1 to –0.7
1.5	0.6 ± 0.4	–0.1 to –0.7	2.5	1.6 ± 0.8	–0.2 to –1.0

All dimensions in mm

Part No.	Nominal size D_i	Nominal size D_o	Width B max. min.	Wall thickness S_3 max. min.	Shaft-ø D_J [d8] max. min.	Housing-ø D_H [H7] max. min.	Bush i-ø $D_{i,a,m}$ machined to [H7] max. min.	Clearance C_D max. min.	Oil hole -ø d_L
MB 1615 DX	16	18	15.25 14.75	1.108 1.082	15.950 15.923	18.018 18.000	16.018 16.000	0.095 0.050	4
MB 1620 DX			20.25 19.75						
MB 1625 DX			25.25 24.75						
MB 1815 DX	18	20	15.25 14.75		17.950 17.923	20.021 20.000	18.018 18.000		
MB 1820 DX			20.25 19.75						
MB 1825 DX			25.25 24.75						
MB 2010 DX	20	23	10.25 9.75	1.608 1.576	19.935 19.902	23.021 23.000	20.021 20.000	0.119 0.065	
MB 2015 DX			15.25 14.75						
MB 2020 DX			20.25 19.75						
MB 2025 DX			25.25 24.75						
MB 2030 DX			30.25 29.75						
MB 2215 DX	22	25	15.25 14.75		21.935 21.902	25.021 25.000	22.021 22.000		6
MB 2220 DX			20.25 19.75						
MB 2225 DX			25.25 24.75						
MB 2230 DX			30.25 29.75						
MB 2415 DX	24	27	15.25 14.75		23.935 23.902	27.021 27.000	24.021 24.000		
MB 2420 DX			20.25 19.75						
MB 2425 DX			25.25 24.75						
MB 2430 DX			30.25 29.75						
MB 2515 DX	25	28	15.25 14.75		24.935 24.902	28.021 28.000	25.021 25.000		
MB 2520 DX			20.25 19.75						
MB 2525 DX			25.25 24.75						
MB 2530 DX			30.25 29.75						
MB 2820 DX	28	32	20.25 19.75	2.108 2.072	27.935 27.902	32.025 32.000	28.021 28.000		
MB 2825 DX			25.25 24.75						
MB 2830 DX			30.25 29.75						
MB 3020 Dx	30	34	20.25 19.75		30.000 29.967	34.025 34.000	30.021 30.000		
MB 3030 DX			30.25 29.75						
MB 3040 DX			40.25 39.75						

10 Standard Products

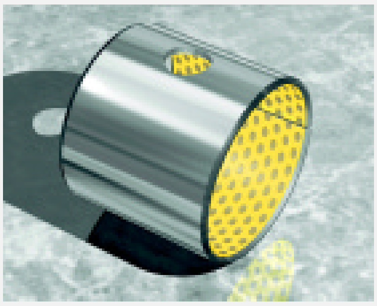
Part No.	Nominal size D_i	Nominal size D_o	Width B max. min.	Wall thickness S_3 max. min.	Shaft-ø D_J [d8] max. min.	Housing-ø D_H [H7] max. min.	Bush i-ø $D_{i,a,m}$ machined to [H7] max. min.	Clearance C_D max. min.	Oil hole -ø d_L
MB 3220 DX	32	36	20.25 19.75	2.108 2.072	31.920 31.881	36.025 36.000	32.025 32.000	0.144 0.080	6
MB 3230 DX	32	36	30.25 29.75	2.108 2.072	31.920 31.881	36.025 36.000	32.025 32.000	0.144 0.080	6
MB 3235 DX	32	36	35.25 34.75	2.108 2.072	31.920 31.881	36.025 36.000	32.025 32.000	0.144 0.080	6
MB 3240 DX	32	36	40.25 39.75	2.108 2.072	31.920 31.881	36.025 36.000	32.025 32.000	0.144 0.080	6
MB 3520 DX	35	39	20.25 19.75	2.108 2.072	34.920 34.881	39.025 39.000	35.025 35.000	0.144 0.080	6
MB 3530 DX	35	39	30.25 29.75	2.108 2.072	34.920 34.881	39.025 39.000	35.025 35.000	0.144 0.080	6
MB 3550 DX	35	39	50.25 49.75	2.108 2.072	34.920 34.881	39.025 39.000	35.025 35.000	0.144 0.080	6
MB 3720 DX	37	41	20.25 19.75	2.108 2.072	36.920 36.881	41.025 41.000	37.025 37.000	0.144 0.080	6
MB 4020 DX	40	44	20.25 19.75	2.108 2.072	39.920 39.881	44.025 44.000	40.025 40.000	0.144 0.080	8
MB 4030 DX	40	44	30.25 29.75	2.108 2.072	39.920 39.881	44.025 44.000	40.025 40.000	0.144 0.080	8
MB 4040 DX	40	44	40.25 39.75	2.108 2.072	39.920 39.881	44.025 44.000	40.025 40.000	0.144 0.080	8
MB 4050 DX	40	44	50.25 49.75	2.108 2.072	39.920 39.881	44.025 44.000	40.025 40.000	0.144 0.080	8
MB 4520 DX	45	50	20.25 19.75	2.634 2.588	44.920 44.881	50.025 50.000	45.025 45.000	0.144 0.080	8
MB 4530 DX	45	50	30.25 29.75	2.634 2.588	44.920 44.881	50.025 50.000	45.025 45.000	0.144 0.080	8
MB 4540 DX	45	50	40.25 39.75	2.634 2.588	44.920 44.881	50.025 50.000	45.025 45.000	0.144 0.080	8
MB 4545 DX	45	50	45.25 44.75	2.634 2.588	44.920 44.881	50.025 50.000	45.025 45.000	0.144 0.080	8
MB 4550 DX	45	50	50.25 49.75	2.634 2.588	44.920 44.881	50.025 50.000	45.025 45.000	0.144 0.080	8
MB 5040 DX	50	55	40.25 39.75	2.634 2.588	49.920 49.881	55.030 55.000	50.025 50.000	0.144 0.080	8
MB 5060 DX	50	55	60.25 59.75	2.634 2.588	49.920 49.881	55.030 55.000	50.025 50.000	0.144 0.080	8
MB 5520 DX	55	60	20.25 19.75	2.634 2.588	54.900 54.854	60.030 60.000	55.030 55.000	0.176 0.100	8
MB 5525 DX	55	60	25.25 24.75	2.634 2.588	54.900 54.854	60.030 60.000	55.030 55.000	0.176 0.100	8
MB 5530 DX	55	60	30.25 29.75	2.634 2.588	54.900 54.854	60.030 60.000	55.030 55.000	0.176 0.100	8
MB 5540 DX	55	60	40.25 39.75	2.634 2.588	54.900 54.854	60.030 60.000	55.030 55.000	0.176 0.100	8
MB 5550 DX	55	60	50.25 49.75	2.634 2.588	54.900 54.854	60.030 60.000	55.030 55.000	0.176 0.100	8
MB 5560 DX	55	60	60.25 59.75	2.634 2.588	54.900 54.854	60.030 60.000	55.030 55.000	0.176 0.100	8
MB 6030 DX	60	65	30.25 29.75	2.634 2.588	59.900 59.854	65.030 65.000	60.030 60.000	0.176 0.100	8
MB 6040 DX	60	65	40.25 39.75	2.634 2.588	59.900 59.854	65.030 65.000	60.030 60.000	0.176 0.100	8
MB 6060 DX	60	65	60.25 59.75	2.634 2.588	59.900 59.854	65.030 65.000	60.030 60.000	0.176 0.100	8
MB 6070 DX	60	65	70.25 69.75	2.634 2.588	59.900 59.854	65.030 65.000	60.030 60.000	0.176 0.100	8

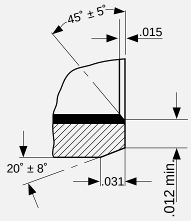
Part No.	Nominal size		Width B	Wall thickness S_3	Shaft-ø D_J [d8]	Housing-ø D_H [H7]	Bush i-ø $D_{i,a,m}$ machined to [H7]	Clearance C_D	Oil hole -ø d_L
	D_i	D_o	max. min.	max. min.	max. min.	max. min.	max. min.	max. min.	
MB 6540 DX	65	70	40.25 39.75	2.634 2.568	64.900 64.854	70.030 70.000	65.030 65.000	0.176 0.100	8
MB 6550 DX			50.25 49.75						
MB 6560 DX			60.25 59.75						
MB 6570 DX			70.25 69.75						
MB 7040 DX	70	75	40.25 39.75		69.900 69.854	75.030 75.000	70.030 70.000		
MB 7050 DX			50.25 49.75						
MB 7065 DX			65.25 64.75						
MB 7070 DX			70.25 69.75						
MB 7080 DX			80.25 79.75						
MB 7540 DX	75	80	40.25 39.75		74.900 74.854	80.030 80.000	75.030 75.000		9.5
MB 7560 DX			60.25 59.75						
MB 7580 DX			80.25 79.75						
MB 8040 DX	80	85	40.50 39.50		79.900 79.854	85.035 85.000	80.030 80.000		
MB 8060 DX			60.50 59.50						
MB 8080 DX			80.50 79.50						
MB 80100 DX			100.50 99.50						
MB 8530 DX	85	90	30.50 29.50		84.880 84.826	90.035 90.000	85.035 85.000	0.209 0.120	
MB 8540 DX			40.50 39.50						
MB 8560 DX			60.50 59.50						
MB 8580 DX			80.50 79.50						
MB 85100 DX			100.50 99.50						
MB 9040 DX	90	95	40.50 39.50		89.880 89.826	95.035 95.000	90.035 90.000		
MB 9060 DX			60.50 59.50						
MB 9090 DX			90.50 89.50						
MB 90100 DX			100.50 99.50						
MB 9560 DX	95	100	60.50 59.50		94.880 94.826	100.035 100.000	95.035 95.000		
MB 95100 DX			100.50 99.50						
MB 10050 DX	100	105	50.50 49.50		99.880 99.826	105.035 105.000	100.035 100.000		
MB 10060 DX			60.50 59.50						
MB 10080 DX			80.50 79.50						
MB 10095 DX			95.50 94.50						
MB 100115DX			115.50 114.50						

10 Standard Products

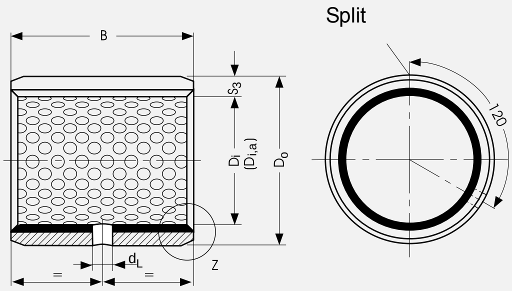
Part No.	Nominal size		Width B	Wall thickness S_3	Shaft-ø D_J [d8]	Housing-ø D_H [H7]	Bush i-ø $D_{i,a,m}$ machined to [H7]	Clearance C_D	Oil hole -ø d_L
	D_i	D_o	max. min.	max. min.	max. min.	max. min.	max. min.	max. min.	
MB 10560 DX	105	110	60.50 59.50	2.634 2.568	104.880 104.826	110.035 110.000	105.035 105.000	0.209 0.120	9.5
MB 105110 DX			110.50 109.50						
MB 105115 DX			115.50 114.50						
MB 11060 DX	110	115	60.50 59.50		109.880 109.826	115.035 115.000	110.035 105.000		
MB 110115 DX			115.50 114.50						
MB 11550 DX	115	120	50.50 49.50		114.880 114.826	120.035 120.000	115.035 115.000		
MB 11570 DX			70.50 69.95						
MB 12060 DX	120	125	60.50 59.50		119.880 119.826	125.040 125.000	120.035 120.000		
MB 120100 DX			100.50 99.50						
MB 125100 DX	125	130	100.50 99.50		124.855 124.792	130.040 130.000	125.040 125.000	0.248 0.145	
MB 13050 DX	130	135	50.50 49.50	2.619 2.564	129.855 129.792	135.040 135.000	130.040 130.000		-
MB 13060 DX			60.50 59.50						
MB 130100 DX			100.50 99.50						
MB 13560 DX	135	140	60.50 59.50		134.855 134.792	140.040 140.000	135.040 135.000		
MB 13580 DX			80.50 79.50						
MB 14060 DX	140	145	60.50 59.50		139.855 139.792	145.040 145.000	140.040 140.000		
MB 140100 DX			100.50 99.50						
MB 15060 DX	150	155	60.50 59.50		149.855 149.792	155.040 155.000	150.040 150.000		
MB 15080 DX			80.50 79.50						
MB 150100 DX			100.50 99.50						

10.3 Inch DX cylindrical bushes





Detail Z



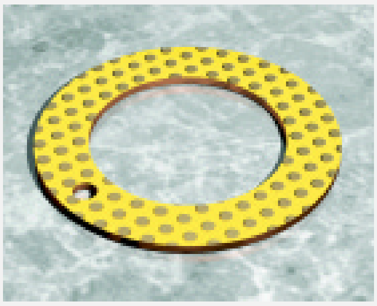
Part Nr.	Nominal size		Wall thickness S_3	Width B ±0.010"	Housing-ø D_H	Shaft-ø D_J	Bush i-ø $D_{i,a}$ as supplied	Shaft for machined bush i-ø D_{Jm}	Bush i-ø $D_{i,a}$ mahined to H 7	Oil hole-ø d_L	Running Clearance C_D as supplied	Running Clearance C_D machined
	D_i	D_o	max. min.	max. min.	max. min.	max. min.	max. min.	max. min.	max. min.		max. min.	max. min.
06DX06				0.385 0.365						No hole		
06DX08	$^{3}/_{8}$	$^{15}/_{32}$		0.510 0.490	0.4694 0.4687	0.3648 0.3639	0.3694 0.3667	0.3734 0.3725	0.3756 0.3750		0.0055 0.0019	0.0031 0.0016
06DX12				0.760 0.740								
07DX08	$^{7}/_{16}$	$^{17}/_{32}$		0.510 0.490	0.5319 0.5312	0.4273 0.4263	0.4319 0.4292	0.4355 0.4345	0.4382 0.4375		0.0056 0.0019	
07DX12				0.760 0.740								
08DX06				0.385 0.365								
08DX08	$^{1}/_{2}$	$^{19}/_{32}$		0.510 0.490	0.5944 0.5937	0.4897 0.4887	0.4944 0.4917	0.4980 0.4970	0.5007 0.5000			
08DX10			0.0510 0.0500	0.635 0.615							0.0057 0.0020	
08DX14				0.885 0.865								
09DX08	$^{9}/_{16}$	$^{21}/_{32}$		0.510 0.490	0.6569 0.6562	0.5522 0.5512	0.5569 0.5542	0.5605 0.5595	0.5632 0.5625			0.0037 0.0020
09DX12				0.760 0.740						$^{5}/_{32}$		
10DX08				0.510 0.490								
10DX10	$^{5}/_{8}$	$^{23}/_{32}$		0.635 0.615	0.7195 0.7187	0.6146 0.6136	0.6195 0.6167	0.6230 0.6220	0.6257 0.6250		0.0059 0.0021	
10DX12				0.760 0.740								
10DX14				0.885 0.865								
11DX14	$^{11}/_{16}$	$^{25}/_{32}$		0.885 0.865	0.7820 0.7812	0.6770 0.6760	0.6820 0.6792	0.6855 0.6845	0.6882 0.6875		0.0060 0.0022	
12DX08				0.510 0.490								
12DX12	$^{3}/_{4}$	$^{7}/_{8}$	0.0669 0.0657	0.760 0.740	0.8758 0.8750	0.7390 0.7378	0.7444 0.7412	0.7475 0.7463	0.7508 0.7500		0.0066 0.0022	0.0045 0.0025
12DX16				1.010 0.990								

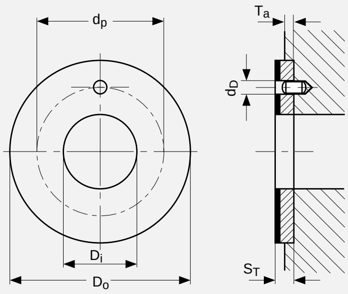
All dimensions in inch

Part Nr.	Nominal size		Wall thick-ness S_3	Width B ±0.010“	Hou-sing-ø D_H	Shaft-ø D_J	Bush i-ø $D_{i,a}$ as supplied	Shaft for ma-chined bush i-ø D_{Jm}	Bush i-ø $D_{i,a}$ mahined to H 7	Oil hole-ø d_L	Running Clea-rance C_D as supplied	Running Clearance C_D machined
	D_i	D_o	max. min.	max. min.	max. min.	max. min.	max. min.	max. min.	max. min.		max. min.	max. min.
14DX12	$^{7}/_{8}$	1	0.0669 0.0657	0.760 0.740	1.0008 1.0000	0.8639 0.8627	0.8694 0.8662	0.8725 0.8713	0.8758 0.8750	$^{1}/_{4}$	0.0067 0.0023	0.0045 0.0025
14DX14				0.885 0.865								
14DX16				1.010 0.990								
16DX12	1	$1^{1}/_{8}$		0.760 0.740	1.1258 1.1250	0.9888 0.9876	0.9944 0.9912	0.9975 0.9963	1.0008 1.0000		0.0068 0.0024	
16DX16				1.010 0.990								
16DX24				1.510 1.490								
18DX12	$1^{1}/_{8}$	$1^{9}/_{32}$	0.0824 0.0810	0.760 0.740	1.2822 1.2812	1.1138 1.1126	1.1202 1.1164	1.1225 1.1213	1.1258 1.2500		0.0076 0.0026	
18DX16				1.010 0.990								
20DX12	$1^{1}/_{4}$	$1^{13}/_{32}$		0.760 0.740	1.4072 1.4062	1.2387 1.2371	1.2452 1.2414	1.2470 1.2454	1.2510 1.2500		0.0081 0.0027	0.0056 0.0030
20DX16				1.010 0.990								
20DX20				1.260 1.240								
20DX28				1.760 1.740								
22DX16	$1^{3}/_{8}$	$1^{17}/_{32}$		1.010 0.990	1.5322 1.5312	1.3635 1.3619	1.3702 1.3664	1.3720 1.3704	1.3760 1.3750		0.0083 0.0029	
22DX22				1.385 0.365								
22DX28				1.760 1.740								
24DX16	$1^{1}/_{2}$	$1^{21}/_{32}$		1.010 0.990	1.6572 1.6562	1.4884 1.4868	1.4952 1.4914	1.4970 1.4954	1.5010 1.5000	$^{5}/_{16}$	0.0084 0.0030	
24DX20				1.260 1.240								
24DX24				1.510 1.490								
24DX32				2.010 1.990								
26DX16	$1^{5}/_{8}$	$1^{25}/_{32}$		1.010 0.990	1.7822 1.7812	1.6133 1.6117	1.6202 1.6164	1.6220 1.6204	1.6260 1.6250		0.0085 0.0031	
26DX24				1.510 1.490								
28DX16	$1^{3}/_{4}$	$1^{15}/_{16}$	0.0980 0.0962	1.010 0.990	1.9385 1.9375	1.7383 1.7367	1.7461 1.7415	1.7470 1.7454	1.7510 1.7500		0.0094 0.0032	
28DX24				1.510 1.490								
28DX28				1.760 1.740								
28DX32				2.010 1.990								
30DX16	$1^{7}/_{8}$	$2^{1}/_{16}$		1.510 1.490	2.0637 2.0625	1.8632 1.8616	1.8713 1.8665	1.8720 1.8704	1.8760 1.8750		0.0097 0.0033	
30DX30				1.885 1.865								
30DX36				2.260 2.240								
32DX16	2	$2^{3}/_{16}$		1.010 0.990	2.1887 2.1875	1.9881 1.9863	1.9963 1.9915	1.9960 1.9942	2.0012 2.0000		0.0100 0.0034	0.0070 0.0040
32DX24				1.510 1.490								
32DX32				2.010 1.990								
32DX40				2.510 2.490								

Part Nr.	Nominal size		Wall thick-ness S_3	Width B ±0.010“	Hou-sing-ø D_H	Shaft-ø D_J	Bush i-ø $D_{i,a}$ as supplied	Shaft for ma-chined bush i-ø D_{Jm}	Bush i-ø $D_{i,a}$ mahined to H 7	Oil hole-ø d_L	Running Clea-rance C_D as supplied	Running Clearanc e C_D machine d
	D_i	D_o	max. min.	max. min.	max. min.	max. min.	max. min.	max. min.	max. min.		max. min.	max. min.
36DX32	$2^{1}/_{4}$	$2^{7}/_{16}$	0.0980 0.0962	2.010 1.990	2.4387 2.4375	2.2378 2.2360	2.2463 2.2415	2.2460 2.2442	2.2512 2.2500	$^{5}/_{16}$	0.0103 0.0037	0.0070 0.0040
36DX36				2.260 2.240								
36DX40				2.510 2.490								
40DX32	$2^{1}/_{2}$	$2^{11}/_{16}$		2.010 1.990	2.6887 2.6875	2.4875 2.4857	2.4963 2.4915	2.4960 2.4942	2.5012 2.5000		0.0106 0.0040	
40DX40				2.510 2.490								
44DX32	$2^{3}/_{4}$	$2^{15}/_{16}$	0.0991 0.0965	2.010 1.990	2.9387 2.9375	2.7351 2.7333	2.7457 2.7393	2.7460 2.7442	2.7512 2.7500		0.0124 0.0042	
44DX40				2.510 2.490								
44DX48				3.010 2.990								
44DX56				3.510 3.490								
48DX32	3	$3^{3}/_{16}$		2.010 1.990	3.1889 3.1875	2.9849 2.9831	2.9959 2.9893	2.9960 2.9942	3.0012 3.0000	$^{3}/_{8}$	0.0128 0.0044	
48DX48				3.010 2.990								
48DX60				3.760 3.740								
56DX40	$3^{1}/_{2}$	$3^{11}/_{16}$		2.510 2.490	3.6889 3.6875	3.4844 3.4822	3.4959 3.4893	3.4950 3.4928	3.5014 3.5000		0.0137 0.0049	0.0086 0.0050
56DX48				3.010 2.990								
56DX60				3.760 3.740								
64DX48	4	$4^{3}/_{16}$		3.010 2.990	4.1889 4.1875	3.9839 3.9817	3.9959 3.9893	3.9950 3.9928	4.0014 4.0000		0.0142 0.0054	
64DX60				3.760 3.740								
64DX76				4.760 4.740								

10.4 DX Thrust Washers - metric

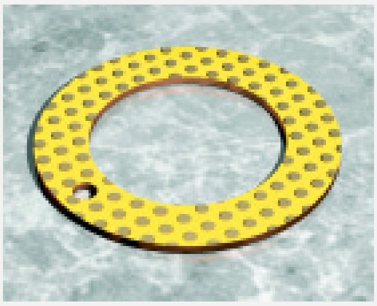


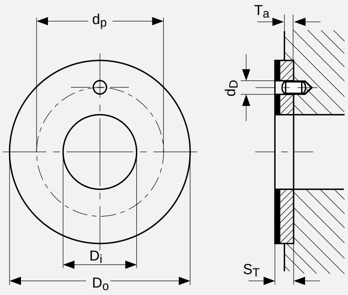


<table>
<tr><th rowspan="2">Part No.</th><th>Inside-ø D_i</th><th>Outside-ø D_o</th><th>Thickness S_T</th><th>Dowel hole PCD-ø d_P</th><th>Dowel hole-ø d_D</th><th>Recess depth T_a</th></tr>
<tr><th>max.
min.</th><th>max.
min.</th><th>max.
min.</th><th>max.
min.</th><th>max.
min.</th><th>max.
min.</th></tr>
<tr><td>WC10DX</td><td>12.25
12.00</td><td>24.00
23.75</td><td rowspan="12">1.577
1.487</td><td>18.12
17.88</td><td>1.875
1.625</td><td rowspan="12">1.20
0.95</td></tr>
<tr><td>WC12DX</td><td>14.25
14.00</td><td>26.00
25.75</td><td>20.12
19.88</td><td rowspan="3">2.375
2.125</td></tr>
<tr><td>WC14DX</td><td>16.25
16.00</td><td>30.00
29.75</td><td>22.12
21.88</td></tr>
<tr><td>WC16DX</td><td>18.25
18.00</td><td>32.00
31.75</td><td>25.12
24.88</td></tr>
<tr><td>WC18DX</td><td>20.25
20.00</td><td>36.00
35.75</td><td>28.12
27.88</td><td rowspan="4">3.375
3.125</td></tr>
<tr><td>WC20DX</td><td>22.25
22.00</td><td>38.00
37.75</td><td>30.12
29.88</td></tr>
<tr><td>WC22DX</td><td>24.25
24.00</td><td>42.00
41.75</td><td>33.12
32.88</td></tr>
<tr><td>WC24DX</td><td>26.25
26.00</td><td>44.00
43.75</td><td>35.12
34.88</td></tr>
<tr><td>WC25DX</td><td>28.25
28.00</td><td>48.00
47.75</td><td>38.12
37.88</td><td rowspan="6">4.375
4.125</td></tr>
<tr><td>WC30DX</td><td>32.25
32.00</td><td>54.00
53.75</td><td>43.12
42.88</td></tr>
<tr><td>WC35DX</td><td>38.25
38.00</td><td>62.00
61.75</td><td>50.12
49.88</td></tr>
<tr><td>WC40DX</td><td>42.25
42.00</td><td>66.00
65.75</td><td>54.12
53.88</td></tr>
<tr><td>WC45DX</td><td>48.25
48.00</td><td>74.00
73.75</td><td rowspan="2">2.600
2.510</td><td>61.12
60.88</td><td rowspan="2">1.70
1.45</td></tr>
<tr><td>WC50DX</td><td>52.25
52.00</td><td>78.00
77.75</td><td>65.12
64.88</td></tr>
</table>

All dimensions in mm

10.5 DX Thrust Washers - inch

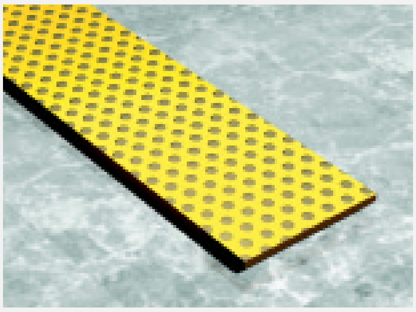


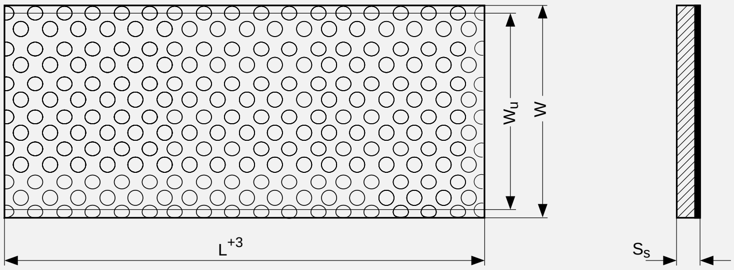


Part No.	Inside-ø D_i	Outside-ø D_o	Thickness S_T	Dowel hole PCD-ø d_P	Dowel hole-ø d_D	Recess depth T_a
	max. min.	max. min.	max. min.	max. min.	max. min.	max. min.
DX06	0.5100 0.5000	0.8750 0.8650	0.0660 0.0625	0.6920 0.6820	0.0770 0.0670	0.0500 0.0400
DX07	0.5720 0.5620	1.0000 0.9900		0.7860 0.7760		
DX08	0.6350 0.6250	1.1250 1.1150		0.8800 0.8700	0.1090 0.0990	
DX09	0.6970 0.6870	1.1870 1.1770		0.9420 0.9320		
DX10	0.7600 0.7500	1.2500 1.2400		1.0050 0.9950		
DX11	0.8220 0.8120	1.3750 1.3650		1.0990 1.0890		
DX12	0.8850 0.8750	1.5000 1.4900		1.1920 1.1820	0.1400 0.1300	
DX14	1.0100 1.0000	1.7500 1.7400		1.3800 1.3700		
DX16	1.1350 1.1250	2.0000 1.9900		1.5670 1.5570	0.1710 0.1610	
DX18	1.2600 1.2500	2.1250 2.1150		1.6920 1.6820		
DX20	1.3850 1.3750	2.2500 2.2400		1.8170 1.8070		
DX22	1.5100 1.5000	2.5000 2.4900		2.0050 1.9950	0.2020 0.1920	
DX24	1.6350 1.6250	2.6250 2.6150		2.1300 2.1200		
DX26	1.7600 1.7500	2.7500 2.7400		2.2550 2.2450		
DX28	2.0100 2.0000	3.0000 2.9900	0.0970 0.0935	2.5050 2.4950		0.0800 0.0700
DX30	2.1350 2.1250	3.1250 3.1150		2.6300 2.6200		
DX32	2.2600 2.2500	3.2500 3.2400		2.7550 2.7450		

All dimensions in inch

10.6 DX Strip - metric





Group No.	Length L	Usable Width W_u	Thickness S_S max. min.
S100 90 DX	500	93	1.07 1.03
S152 00 DX		200	1.56 1.52
S202 00 DX		218	2.05 2.01
S252 00 DX			2.57 2.52

All dimensions in mm

10.7 DX Strip - inch

Group No.	Length L	Usable Width W_u	Thickness S_S max. min.
B	18	2.75	0.0492 0.0480
C		4	0.0642 0.0630
D			0.0795 0.0783
E			0.0949 0.0937

All dimensions in inch

This handbook was designed by
Profidoc Silvia Freitag

www.profidoc.de