The PumPac 8000 Series is used in centrifugal pumps, large vertical electric motors, compressors, centrifuges, and other applications subject to thrust loads operating at relatively high speeds. The bearings are mounted so that the 40 degree bearing takes the primary thrust, or axial, load.

Traditionally, matched sets of 40 degree angular contact bearings are used to obtain maximum theoretical fatigue life, but in most instances only a fraction of the calculated life is actually achieved. At the heart of these premature failures are phenomena known as "ball sliding" and "ball shuttling" in the unloaded or inactive bearing. Angular contact bearings used in high speed (e.g. 3600 RPM) pumps and other applications require a minimum axial load for proper operation. Without axial load, gyroscopic forces in the unloaded bearing can cause the balls to rotate perpendicular to their true rolling axis and momentarily lose contact with the raceway. As a result, a microscopic wear or lapping process occurs giving the appearance of a burnished or polished raceway. The oil film thickness separating the ball and raceway is reduced producing friction and heat with lower oil viscosity, accelerating wear. This thermally unstable condition dramatically reduces bearing service life.

The main benefit of the PumPac is that the 15 degree bearing is designed with considerably less internal clearance than the 40 degree bearing making it less susceptible to the gyroscopic forces which result in ball sliding or shuttling. This bearing also provides additional radial stiffness helping to maintain integrity of the shaft and mechanical seals. The 40 degree, loaded bearing provides sufficient axial rigidity under the imposed thrust load. The PumPac is furnished with a one piece, inner ring land guided, machined bronze cage and is manufactured to meet ABEC-3 grade tolerances. Dimensional stability is assured by heat treating for operation up to $375^{\circ}F$ ($190^{\circ}C$).

For identification purposes the 40 degree bearing is marked with the letter "A" and the 15 degree bearing with the letter "B". A "V" is etched on the outer ring surface of the set so that the apex falls on the "B" bearing. The set should be mounted so that the "V" points in the same direction as the primary shaft thrust load which places this load on bearing "A". Any reversing load is carried on bearing "B".

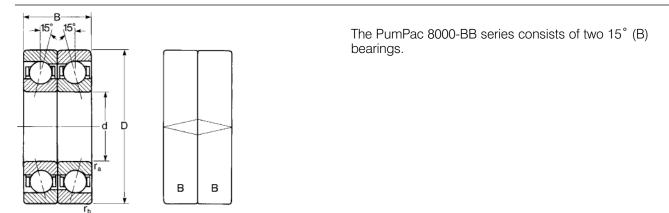
The PumPac is also available in the 8000BB PumPac Diamond series, which consists of a matched set of 15 degree (B) bearings in a back-to-back arrangement. The marking on the outer ring surface forms a diamond. This series is desirable in high speed, lightly loaded applications in which axial loads are balanced, resulting in reduced operating temperatures and increased life.

For applications involving very heavy primary thrust loads the PumPac may be furnished as a triplex set identified as the 8000AAB Series. It consists of two 40 degree bearings in tandem matched back-to-back (DB) with one 15 degree bearing. The outer ring surface of the set is marked with a "V" pointing in the direction of primary thrust.

For applications having a rotating shaft and stationary housing, an ISO k5 shaft fit and an ISO H6 housing fit is recommended as shown in the table on page 180.

NOTE: The basic radial load rating, C, is calculated according to the actual bearing geometry and not according to ISO/ABMA Standards.

PumPac Diamond 8000-BB Series



ABEC 3 Tolerances

											Basic Radial Load Rating ²⁾				Speed Rating ³⁾	
MRC Bearing Number	Bore <u>d</u> mm	in	Outsi Diam D mm		Width <u>B</u> mm	in	r <u>a</u> mm	Fillet R	adius ¹⁾ <u>r_b</u> mm	in		amic ³⁴⁾ Ibf		atic C _o	Grease RPM	Oil RPM
8218BB	90	3.5433	160	6.2992	60	2.3622	2.0	.08	1.0	.04	203 000	45 600	216 000	48 600	3 800	5 400
8219BB	95	3.7402	170	6.6929	64	2.5197	2.0	.08	1.0	.04	216 000	48 600	236 000	53 100	3 600	5 100
8220BB	100	3.9370	180	7.0866	68	2.6772	2.0	.08	1.0	.04	238 000	53 500	270 000	60 700	3 400	4 800
8222BB	110	4.3307	200	7.8740	70	2.9921	2.0	.08	1.0	.04	296 000	66 500	340 000	76 400	3 000	4 400
8224BB	120	4.7244	215	8.4646	80	3.1496	2.0	.08	1.0	.04	319 000	71 800	390 000	87 700	2 800	4 100
8238BB	190	7.4803	340	13.3858	110	4.3307	3.0	.12	1.0	.04	605 000	136 000	1 000 000	225 000	1 800	2 600
8308BB	40	1.5748	90	3.5433	46	1.8110	1.5	.06	1.0	.04	79 300	17 800	67 000	15 100	7 500	11 000
8309BB	45	1.7717	100	3.9370	50	1.9685	1.5	.06	1.0	.04	95 600	21 500	81 500	18 300	6 800	9 600
8310BB	50	1.9685	110	4.3307	54	2.1260	2.0	.08	1.0	.04	124 000	27 900	104 000	23 400	6 300	9 000
8311BB	55	2.1654	120	4.7244	58	2.2845	2.0	.08	1.0	.04	143 000	32 100	122 000	27 400	5 600	8 100
8312BB	60	2.3622	130	5.1181	62	2.4409	2.0	.08	1.0	.04	165 000	37 100	143 000	32 100	5 000	7 200
8313BB	65	2.5591	140	5.5118	66	2.5984	2.0	.08	1.0	.04	174 000	39 100	160 000	36 000	4 800	6 800
8314BB	70	2.7559	150	5.9055	70	2.7559	2.0	.08	1.0	.04	199 000	44 700	190 000	42 700	4 500	6 300
8315BB	75	2.9528	160	6.2992	74	2.9134	2.0	.08	1.0	.04	238 000	53 500	228 000	51 300	4 300	6 000
8316BB	80	3.1496	170	6.6929	78	3.0709	2.0	.08	1.0	.04	255 000	57 300	260 000	58 500	3 800	5 400
8317BB	85	3.3465	180	7.0866	82	3.2283	2.5	.10	1.0	.04	281 000	63 200	260 000	58 500	3 600	5 100
8318BB	90	3.5433	190	7.4803	86	3.3858	2.5	.10	1.0	.04	302 000	67 900	325 000	73 100	3 400	4 800
8319BB	95	3.7402	200	7.8740	90	3.5433	2.5	.10	1.0	.04	325 000	73 100	360 000	80 900	3 300	4 500
8320BB	100	3.9370	215	8.4646	94	3.7008	2.5	.10	1.0	.04	345 000	77 600	400 000	89 900	3 000	4 400
8322BB	110	4.3307	240	9.4488	100	3.9370	2.5	.10	1.0	.04	416 000	93 500	510 000	115 000	2 600	3 900
8326BB	130	5.1181	280	11.0236	116	4.5669	3.0	.12	1.0	.04	475 000	107 000	695 000	156 000	2 300	3 300
8330BB	150	5.9055	320	12.5984	130	5.1181	3.0	.12	1.0	.04	624 000	140 000	950 000	214 000	2 000	2 900
8336BB	180	7.0866	380	14.9606	150	5.9055	3.0	.12	1.0	.04	780 000	175 000	1 270 000	286 000	1 600	2 400

¹⁾ Fillet radius indicates maximum fillet radius on shaft or in housing which bearing corner will clear.
²⁾ For thrust rating multiply C by 0.51 and Co by 1.00.
³⁾ Values have been determined through historical application and practice. For a more complete explanation, see page 276.

⁴⁾ Rating for one million revolutions or 500 hours at 331/3 RPM.

The following method considers only thrust load in either direction with negligible radial load. For combined radial and thrust loads consult MRC Technical Services for analysis.

Dynamic equivalent radial load

Primary thrust on bearing A (40°) $P = 0.57 F_A$

Reversing thrust on bearing B (15°)

- $P = YF_A$
- P = Dynamic equivalent radial load
- $F_A = Thrust load$
- Y = Thrust load factor
- Z = Number of balls
- D = Ball diameter

	Y		
N/mm	Lb/in.		
0.172	25	1.47	
0.345	50	1.40	
0.689	100	1.30	
1.03	150	1.23	
1.38	200	1.19	
2.07	300	1.12	
3.45	500	1.02	
5.17	750	1.00	
6.89	1 000	1.00	

	ZI	D 2		ZI)²		ZD ²	
Size			Size			Size		
	mm	inch		mm	inch		mm	inch
8218B	7410	11.5	8310B	3990	6.19	8318B	11800	18.3
8219B	7900	12.3	8311B	4690	7.26	8319B	13100	20.3
8220B	9070	14.1	8312B	5430	8.42	8320B	14400	22.4
8222B	11700	18.1	8313B	5930	9.19	8322B	18900	29.3
8224B	13100	20.3	8314B	6800	10.5	8326B	25700	39.8
8238B	31200	48.4	8315B	8390	13.0	8330B	31500	48.8
8308B	2450	3.80	8316B	9470	14.7	8336B	45700	70.9
8309B	3020	4.69	8317B	10600	16.5			

Values of Y for loads not shown are obtained from chart at right.

Notes: For 8000BB series see page 80. For combined radial and thrust loads please contact MRC bearing services.

Life rating

L10 =
$$\left(\frac{C}{P}\right)^3$$
 (millions of revolutions)

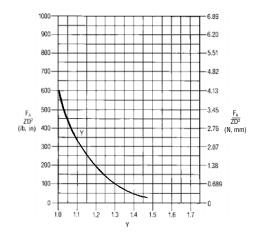
or

$$L10h = \frac{10^6}{60n} \left(\frac{C}{P}\right)^3 \quad (Hours)$$

- C = Basic dynamic load rating
- P = Dynamic equivalent radial load
- n = speed in RPM

Static equivalent radial load

Bearing Å (40°) $P_0 = 0.26 F_A$ Bearing B (15°) $P_0 = 0.5 F_A$ $P_0 = Static equivalent radial load$ $<math>F_A = Thrust load$





Dynamic equivalent radial load and life calculation examples PumPac 8000 Series

Bearing size: 8310 Speed: 3500 RPM Basic dynamic radial load rating (C): Bearing A = 19600 lbf Bearing B = 17100 lbf

Bearing A

Primary thrust load (F_A) = 3000 Equivalent load (P) = 0.57 F_A P = 0.57 × 3000 = 1710 Life (L10) = $\left(\frac{C}{P}\right)^3 = \left(\frac{19600}{1710}\right)^3 = 1506 \times 10^6$ Rev. or Life (L10h) = $\frac{10^6}{60n} \left(\frac{C}{P}\right)^3 = \frac{10^6}{60 \times 3500} \left(\frac{19600}{1710}\right)^3$ = 7170 Hrs

Bearing B

 $\begin{array}{l} \mbox{Reversing thrust load } (F_{A}) = 1000 \\ F_{A}/ZD^{2} = 1000/6.19 = 162 \\ \mbox{Equivalent load } (P) = Y \ F_{A} \\ Y = 1.22 \\ P = 1.22 \times 1000 = 1220 \\ \mbox{Life } (L10) = \left(\frac{C}{P}\right)^{3} = \left(\frac{17100}{1220}\right)^{3} = 2753 \times 10^{6} \ \mbox{Rev.} \\ \mbox{or} \\ \mbox{Life } (L10h) = \frac{10^{6}}{60n} \left(\frac{C}{P}\right)^{3} = \frac{10^{6}}{60 \times 3500} \left(\frac{17100}{1220}\right)^{3} \\ = 13113 \ \mbox{Hrs} \end{array}$

PumPac Diamond 8000 BB Series Bearing size: 8317BB Speed: 3000 RPM Basic dynamic radial load rating (C): (Single Bearing) = $\frac{C}{(2)^{0.7}} = \frac{63200}{1.625} = 38890$ lbf **Primary Thrust Bearing** Primary thrust load $(F_A) = 2500$ $F_A/ZD^2 = 2500/16.5 = 152$ Equivalent load (P) = $Y F_A$ Y = 1.23 $P = 1.23 \times 2500 = 3075$ Life (L10) = $\left(\frac{C}{P}\right)^3 = \left(\frac{38890}{3075}\right)^3 = 2023 \times 10^6$ Rev. or or Life (L10h) = $\frac{10^6}{60n} \left(\frac{C}{P}\right)^3 = \frac{10^6}{60 \times 3000} \left(\frac{38890}{3075}\right)^3$ = 11238 Hrs **Reversing Thrust Bearing** Reversing thrust load $(F_A) = 1000$ $F_A/ZD^2 = 1000/16.5 = 61$ Equivalent load (P) = $Y F_A$ Y = 1.38P = 1.238 × 1000 = 1380 Life (L10) = $\left(\frac{C}{P}\right)^3 = \left(\frac{38890}{1380}\right)^3$ 381×10^6 Rev.

or
Life (L10h) =
$$\frac{10^6}{60n} \left(\frac{C}{P}\right)^3 = \frac{10^6}{60 \times 3000} \left(\frac{38890}{1380}\right)^3$$

= 124,340 Hrs

PumPac 8000 AAB Series

Bearing size: 8314 AAB Speed: 3500 RPM Basic dynamic radial load rating (C): Bearings AA = 54400 lbf Bearings AA Primary thrust load (F_A) = 5000 Equivalent load (P) = 0.57 F_A P = 0.57 × 5000 = 2850 Life (L10) = $\left(\frac{C}{P}\right)^3 = \left(\frac{54400}{2850}\right)^3 = 6954 \times 10^6$ Rev. or Life (L10h) = $\frac{10^6}{60n} \left(\frac{C}{P}\right)^3 = \frac{10^6}{60 \times 3500} \left(\frac{54400}{2850}\right)^3$ = 33116 Hrs

Bearing B Reversing thrust load $(F_A) = 1500$ $F_A/ZD^2 = 1500/10.5 = 143$ Equivalent load (P) = Y F_A Y = 1.24 P = 1.24 × 1500 = 1860 Life (L10) = $\left(\frac{C}{P}\right)^3 = \left(\frac{27200}{1860}\right)^3 = 3127 \times 10^6$ Rev. or Life (L10h) = $\frac{10^6}{60n} \left(\frac{C}{P}\right)^3 = \frac{10^6}{60 \times 3500} \left(\frac{27200}{1860}\right)^3$ = 14900 Hrs